

UNIVERSITI PUTRA MALAYSIA

COMPUTATIONAL FLUID DYNAMICS ANALYSIS ON THERMAL PERFORMANCE OF SOLAR AIR HEATER WITH INCLINED FINS

BOOTAN SHERZAD QADER

FK 2018 126



COMPUTATIONAL FLUID DYNAMICS ANALYSIS ON THERMAL PERFORMANCE OF SOLAR AIR HEATER WITH INCLINED FINS



Thesis Submitted to the School of Graduate Studies, Universiti Putra Malaysia, in Fulfillment of the Requirements for the Degree of Master of Science

COPYRIGHT

All material contained within the thesis, including without limitation text, logos, icons, photographs, and all other artwork, is copyright material of Universiti Putra Malaysia unless otherwise stated. Use may be made of any material contained within the thesis for non-commercial purposes from the copyright holder. Commercial use of material may only be made with the express, prior, written permission of Universiti Putra Malaysia.

Copyright © Universiti Putra Malaysia



DEDICATION

This thesis is dedicated to all people around me especially for those who involved and contributed a lot of expenses in completing the thesis. First, I dedicate this work to my beloved wife and my family. Second, to my supervisory committee Dr. Eris Elianddy Bin Supeni, Professor Ir. Dr. Khairol Anuar bin Mohd Ariffin and Associate Professor Ir. Dr. Abd. Rahim Abu Talib who have guided me during the projects period. Lastly, I dedicate this thesis to all my friends who gave me their real friendship and advices, and motivated me to complete this thesis.



Abstract of thesis presented to the Senate of Universiti Putra Malaysia in fulfilment of the requirement for the degree of Master of Science

COMPUTATIONAL FLUID DYNAMICS ANALYSIS ON THERMAL PERFORMANCE OF SOLAR AIR HEATER WITH INCLINED FINS

By

BOOTAN SHERZAD QADER

July 2018

Chairman : Eris Elianddy Bin Supeni, PhD

Faculty : Engineering

The depletion of fossil fuel reserves and rise in price instability of fossil fuels and also the impacts of global warming have accelerated the development of renewable energy sources as a reliable alternative energy source. The solar energy is a clean, sustainable, abundant and renewable energy source that has attracted the interest of various researchers across the world for solar energy with related applications. Similarly, solar irradiance can be absorbed by the solar collectors from the sun and thereafter, it is converted to convenient heat needed for heating air. The design of solar air heater is simple and commonly used in different areas of application. The thermal performance of the conventional solar air heater is found to be poor due to low convective heat transfer coefficient between heat collecting surface and working fluid. The main objective of this research is to increase the convection heat transfer coefficient in order to increase the performance of thermal system. In this study, a numerical evaluation was carried out on the heat transfer and the flow friction processes in a solar air heater coupled with inclined fins located underneath the absorber plate. With the constant heat flux application, the average Nusselt number and friction factor as well as the thermo-hydraulic performance parameter (THPP) were comprehensively investigated. The research covered various length of fin in the range of 1.5-2.5 mm, different slant angle (α) of fin in the range of 30°-60°, different pitch (P) of fin in the range of 15-25 mm, and a range of 4,000-24,000 for the Reynolds numbers. For the current computational fluid dynamic (CFD) evaluation, (ANSYS FLUENT v16.1) with renormalization group $k - \varepsilon$ turbulence model is selected for analysis computational domain.

In general, a significant improvement of the heat transfer in a solar air heater having inclined fins has been achieved. The maximum value of 174.05 was observed for the average Nusselt number which corresponds to a length of fin e=2.5mm, slant angle

 α =60° and pitch P=15mm at a Reynolds number of (24,000) for the of parameter range investigated. While, the maximum friction factor of 0.046 was observed which corresponds to a length of fin e=2.5mm, slant angle α =60° and pitch P=15mm at a Reynolds number of (4,000).

Moreover, the quadratic model created by the response surface methodology (RSM) for computation of the THPP was found so that it is applied to a high degree of accuracy to quantify the operating performance of the solar air heater. Based on results of the model, the optimized values of design parameters for the optimal operation of solar air heater to provide the optimal THPP of 1.928 were found to be; length of fin = 1.52 mm, the pitch of fin = 19.04 mm, slant angle = 49° and Reynolds number of 18243.5. According to the optimized values of design parameters, the enhancement ratio of Nusselt number, friction factor and the THPP were found to be 2.53, 2.22 and 1.928, respectively. These correspond to the improvement of the Nusselt number by 153%, the friction factor by 122% as well as the THPP by 92.8% compared to the smooth duct. Finally, the thermal performance of the proposed inclined fin in terms of THPP was compared to other roughness geometries, such as circle (THPP=1.65), square-sectioned (THPP=1.80) and L-shaped (THPP=1.90). Accordingly, a better THPP of 1.928 was observed for the current study.

Abstrak tesis yang dikemukakan kepada Senat Universiti Putra Malaysia sebagai memenuhi keperluan untuk ijazah Master Sains

ANALISIS DINAMIK CECAIR PENGIRAAN TERHADAP PRESTASI TERMA PEMANAS UDARA SURIA DENGAN SIRIP CONDONG

Oleh

BOOTAN SHERZAD QADER

Julai 2018

Pengerusi : Eris Elianddy Bin Supeni, PhD

Fakulti : Kejuruteraan

Penurunan rizab bahan api fosil dan ketidakstabilan harga bahan api fosil dan juga kesan pemanasan global telah mempercepatkan pembangunan sumber tenaga boleh diperbaharui sebagai punca tenaga alternatif yang boleh diharap. Tenaga suria ialah suatu tenaga yang bersih, mampan, banyak dan boleh diperbaharui yang telah menarik minat ramai penyelidik di seluruh dunia untuk mengkaji tenaga suria dan penggunaannya. Sinaran suria contohnya, boleh diserap oleh pengumpul suria dan sesudah itu, ditukar kepada haba sesuai yang diperlukan untuk memanaskan udara. Reka bentuk sebuah pemanas udara adalah mudah menyebabkannya banyak digunakan dalam pelbagai bidang kegunaan. Bagaimanapun, kecekapan terma pemanas udara suria biasa agak rendah disebabkan pekali pemindahan haba olakan yang rendah antara permukaan pengumpul haba dan bendalir kerja. Objektif utama bagi penyelidikan ini adalah untuk meningkatkan pekali pemindahan haba olakan bagi meningkatkan prestasi sistem terma. Dalam kajian ini, penilaian berangka dijalankan ke atas proses pemindahan haba dan aliran geseran dalam sebuah pemanas udara suria yang dipasang dengan sirip condong di bawah plat penyerap. Dengan penerapan fluks haba malar, nombor Nusselt purata dan faktor geseran serta parameter prestasi terma-hidraulik (THPP) diselidik dengan menyeluruh. Penyelidikan ini meliputi pelbagai panjang sirip 1.5-2.5 mm, sudut condong sirip (α) berbeza 30°-60°, junaman sirip (P) berbeza 15-25 mm, dan julat 4,000-24,000 bagi nombor Reynolds. Bagi penilaian Dinamik Cecair Pengiraan (CFD) ini, model gelora (ANSYS FLUENT v16.1) dengan kumpulan penormalan $k - \varepsilon$ dipilih untuk domain pengiraan analisis.

Secara umum, penambahbaikan yang signifikan telah dicapai bagi pemindahan haba dalam sebuah pemanas udara suria dengan sirip condong. Nilai maksimum 174.05 telah diperolehi bagi nombor Nusselt purata bagi panjang sirip e=2.5 mm, sudut

condong sirip α =60° and junaman sirip P=15 mm pada nombor Reynolds (24,000) untuk julat parameter yang dikaji. Sementara itu, faktor geseran maksimum 0.046 pula diperolehi bagi panjang sirip e=2.5 mm, sudut condong sirip α =60° and junaman sirip P=15 mm pada nombor Reynolds (4,000).

Tambahan lagi, model kuadratik yang dicipta RSM untuk pengiraan THPP dicari supaya dapat digunakan dengan darjah kejituan yang tinggi bagi mengukur prestasi kendalian pemanas udara suria tersebut. Berdasarkan keputusan model, nilai optimum parameter reka bentuk bagi kendalian optimum pemanas udara suria untuk mendapat nilai optimum THPP 1.928 ialah; panjang sirip = 1.52 mm, junaman sirip = 19.04 mm, sudut condong = 49° dan nombor Reynolds = 18243.5. Berdasarkan nilai optimum parameter rekaan, catuan penambahan bagi nombor Nusselt, faktor geseran, dan THPP didapati sebanyak 2.53, 2.22, dan 1.928 masing-masing, iaitu bersamaan dengan meningkatkan nombor Nusselt, faktor geseran, dan THPP sebanyak 153%, 122%, dan 92.8% berbanding saluran licin. Akhir sekali, prestasi terma bagi sirip condong yang dicadangkan berdasarkan THPP telah dibandingkan dengan geometri tidak rata lain seperti bulatan (THPP=1.65), seksyen empat segi (THPP=1.80) dan bentuk-L (THPP=1.90), di mana THPP lebih baik bagi kajian ini dapat diperhatikan iaitu 1.928.

ACKNOWLEDGEMENTS

In the name of Allah, the Most Beneficent, the Most Merciful

I would like to express my sincere gratitude to my supervisor, Dr. Eris Elianddy Bin Supeni who continuously monitored progress of this project and channelized my efforts in the best possible way. He has always been enthusiastic, encouraging, and supportive. This work would not be possible without the help of him.

I would like to greatly thank my supervisory committee Professor Ir. Dr. Khairol Anuar bin Mohd Ariffin and Associate Professor Ir. Dr. Abd. Rahim Abu Talib, for their support, the scientific guidance, and the encouragement. Your time and service is sincerely appreciated, and your insights and input to my work have been valuable.

I would like to thank my supportive and loving wife, who has made this life journey to Malaysia and with the Master thesis much more pleasant. Her love and helpful spirit have motivated me to achievements beyond my own expectations.

I would like to thank all my friends for their continuous support that helped me overcome some difficult moments. I wish to express my sincere gratitude to my parents, my parents-in-law who have supported me during my life stages and pray to ALLAH to save them.

I am truly grateful to all colleagues at the Engineering Faculty, Universiti Putra Malaysia for their advice and support.

Thanks go to directorate engineering of Parks-Erbil, Ministry of municipality and tourism who given me the opportunity to gain higher education standard.

This thesis was submitted to the Senate of the Universiti Putra Malaysia and has been accepted as fulfilment of the requirement for the degree of Master of Science. The members of the Supervisory Committee were as follows:

Eris Elianddy Bin Supeni, PhD

Senior Lecturer Faculty of Engineering Universiti Putra Malaysia (Chairman)

Mohd Khairol Anuar bin Mohd Ariffin, PhD

Professor Ir.
Faculty of Engineering
Universiti Putra Malaysia
(Member)

Abd. Rahim Abu Talib, PhD PE

Associate Professor Ir. Faculty of Engineering Universiti Putra Malaysia (Member)

ROBIAH BINTI YUNUS, PhD

Professor and Dean School of Graduate Studies Universiti Putra Malaysia

Date:

Declaration by graduate student

I hereby confirm that:

- this thesis is my original work;
- quotations, illustrations and citations have been duly referenced;
- this thesis has not been submitted previously or concurrently for any other degree at any institutions;
- intellectual property from the thesis and copyright of thesis are fully-owned by Universiti Putra Malaysia, as according to the Universiti Putra Malaysia (Research) Rules 2012;
- written permission must be obtained from supervisor and the office of Deputy Vice-Chancellor (Research and innovation) before thesis is published (in the form of written, printed or in electronic form) including books, journals, modules, proceedings, popular writings, seminar papers, manuscripts, posters, reports, lecture notes, learning modules or any other materials as stated in the Universiti Putra Malaysia (Research) Rules 2012;
- there is no plagiarism or data falsification/fabrication in the thesis, and scholarly integrity is upheld as according to the Universiti Putra Malaysia (Graduate Studies) Rules 2003 (Revision 2012-2013) and the Universiti Putra Malaysia (Research) Rules 2012. The thesis has undergone plagiarism detection software

Signature:	Date:
Name and Matric No: Bo	otan Sherzad Qader, GS45642

Declaration by Members of Supervisory Committee

This is to confirm that:

- the research conducted and the writing of this thesis was under our supervision;
- supervision responsibilities as stated in the Universiti Putra Malaysia (Graduate Studies) Rules 2003 (Revision 2012-2013) were adhered to.

Signature:	
Name of Chairman	
of Supervisory	
Committee:	Dr. Eris Elianddy Bin Supeni
Signature:	**************************************
Name of Member of Supervisory	
Committee:	Professor Dr. Mohd Khairol Anuar bin Mohd Ariffin
Signature:	
Name of Member	
of Supervisory	
Committee:	Associate Professor Dr. Abd. Rahim Abu Talib

TABLE OF CONTENTS

				1	Page
ABS	STRACT	1			i
ABS	TRAK				iii
ACI	KNOWL	EDGE	MENTS		v
APP	PROVAI				vi
DEC	CLARA	ΓION			viii
LIS	T OF TA	BLES			xiii
LIS	T OF FI	GURES	5		xiv
LIS	T OF NO)MEN(CLATURI	E AND ABBREVIATIONS	xix
CHA	APTER				
1	INTR	ODUC	TION		1
	1.1	Backg	ground of t	h <mark>e Study</mark>	1
	1.2	_	em Statem		2
	1.3	Objec	tives		2 3 3
	1.4		of the Stu	ıdy	3
	1.5		s Structure		3
2	LITE	RATU	RE REVI	EW	5
	2.1	Overv			5
	2.2	Solar	Collectors		6
				te Solar Collectors	6
				Plate Solar Collectors	7
	2.3	Classi	fication of	f Solar Air Collectors	8
	2.4		Air Heater		8
		2.4.1	Solar Ai	r Heaters with Thermal Storage	9
		2.4.2		r Heaters without Thermal Storage	13
	2.5			hancement Methods for Solar Air Heaters	19
		2.5.1		ment of Intensity of Solar Incident Radiation on	
			Solar Co		19
		2.5.2	Reduction	on of Thermal Losses	19
				Decrease in Radiative Heat Losses and	
				Convective	20
			2.5.2.2	Using Vacuum in the Gap Space or Alternate	
				Medium	20
			2.5.2.3	Using Selective Absorber Surfaces	20
		2.5.3		ssage Modifications for Convective Heat Transfer	
		2.0.0		ent Enhancement	20
			2.5.3.1	Providing Packing in Air Heating Solar	_0
				Collectors	21
			2.5.3.2	Providing Corrugated Absorber Plate in Air	
			2.0.0.2	Heating Solar Collectors	21
			2.5.3.3	Providing Artificial Roughness on the	4 1
			2.2.3.3	Absorber Plate in Air Heating Solar Collectors	21

	2.6	Artificially Roughened Solar Air Heater Ducts	22
		2.6.1 Different Rib Geometries used and their Effects in Solar	
		Air Heater	22
		2.6.2 Use of Different Fin Geometries and Their Impact on	
		the Solar Air Heater	27
	2.7	CFD Applications in Various Aspects of Solar Air Heaters	32
	2.8	Summary	37
3		HODOLOGY	39
	3.1	Overview	39
	3.2	63	40
	3.3	Define the Parameters and Problem Description	41
	3.4	CFD Simulation	42
		3.4.1 Computational Domain	44
			45
			45
			48
		3.4.5 Selection and Validation of Appropriate Turbulence Model	48
		3.4.6 Convergence criteria	49
		3.4.7 Solution Method	50
	3.5	Grid Independence Test (GIT)	51
	3.6	Code Validation	52
	3.7	Response Surface Methodology (RSM)	53
	3.8	Summary	57
	DEGI	THE AND DISCUSSION	
4		JLTS AND DISCUSSION	58
	4.1	Overview	58
	4.2	Heat Transfer Characteristics	58
		4.2.1 Effect of Length of Fin (e)	58
			60
		4.2.3 Effect of Pitch of Fin (P)	66
	4.3	Friction Characteristics	68
		4.3.1 Effect of Length of Fin (e)	68
		4.3.2 Effect of Slant Angle of Fin (α)	70
		4.3.3 Effect of Pitch of Fin (P)	76
	4.4	Heat Transfer and Friction Factor Enhancement	78
	4.5	Thermo-Hydraulic Performance Parameter (THPP)	80
	4.6	Optimization Using Design of Experiments (DOE)	82
		4.6.1 ANOVA Analysis	82
		4.6.2 Effect of Reynolds Number, Length, Angle and Pitch of	
		Fins on THPP	86
		4.6.3 Optimization of Designing Parameters	90
	4.7	Comparison with Previous Works	92
	4.8	Summary	94
5	CON	CLUSION AND RECOMMENDATIONS	95
	5.1	Conclusions	95
	5.2	Recommendations for Future Work	96

REFERENCES	98
BIODATA OF STUDENT	105
LIST OF PUBLICATIONS	106



LIST OF TABLES

Table		Page
3.1	Geometrical parameters used in solar air heater duct with inclined fin	44
3.2	Thermo-physical properties of air and absorber plate for CFD analysis	44
3.3	Grid independence test for CFD analysis	51
3.4	Factors and corresponding response from CFD analysis	56
4.1	Nusselt number enhancement ratio for different values of length (e), slant angle (α) and pitch (P)	78
4.2	Friction factor enhancement ratio for different values of length (e), slant angle (α) and pitch (P)	79
4.3	Thermo-hydraulic performance parameter (THPP) for different values of length (e), slant angle (α) and pitch (P)	81
4.4	ANOVA table for the THPP (before elimination)	83
4.5	ANOVA table for the THPP (after backward elimination)	84
4.6	Range of input parameters and responses for optimization	90
4.7	Optimum values of input parameters and the response	90
4.8	Nusselt number, friction factor and THPP enhancement ratio for optimized design parameters	92
4.9	Comparison of THPP between CFD result and RSM result for the optimum design of SAH	92
4.10	Comparison between present study and previous work results	93

LIST OF FIGURES

Figur	e	Page
2.1	Bare plate solar collector	7
2.2	Covered plate solar collector	7
2.3	Classification of collectors	8
2.4	Solar air heater integrated with PCM	10
2.5	Cross-sectional view of the collector assembly	11
2.6	Schematic views of greenhouse heating system with thermal energy storage unit	12
2.7	Schematic of the solar air collector with PCM cylinders	12
2.8	Schematic view of the SAH	13
2.9	Variations of temperature rise and efficiency of the collector with mas flow rate for single and double glazing	ss 14
2.10	Construction details of the test air heaters	15
2.11	An actual panel of solar air heater	16
2.12	Schematic assembly of the SAH system	17
2.13	Experimental setup and location of thermocouples	18
2.14	Integral chamfered ribs	22
2.15	V-shaped rib roughness	23
2.16	Wedge shaped transverse integral ribs	23
2.17	Metal grit rib roughness	24
2.18	Arc shaped rib roughness	24
2.19	Discrete W-shaped rib roughness	25
2.20	Discrete V-down rib	25
2.21	Dimple shaped rib roughness	26

2.22	Broken V-rib roughness combined with staggered ribs	26
2.23	Single & Double Pass SAH Showing, (a) Figure of the Practical Rig, (b) Lowers Pass Channel, (c) Upper Pass Channel 28	
2.24	Diagram of the Tested Heaters	29
2.25	View of an Upward-Type Internally-Recycle Flat-Plate SAH with 30	
2.26	A solar box with and without fins	31
2.27	Sectional view of the present model	31
2.28	Solar air heater with herring bone corrugated fins with geometrical description	32
2.29	Roughened absorber plate with different arrangement of circular sectioned transverse rib	34
2.30	Roughened absorber plate with different arrangement of square- sectioned transverse ribs	35
2.31	Roughened absorber plate with different arrangement of L-shaped transverse ribs	36
2.32	Roughened absorber plate with hyperbolic ribs	37
3.1	Methodology flow chart	40
3.2	Schematic diagram (a) 2D of solar air heater (b) section of absorber plate with inclined fins	41
3.3	Flowchart of CFD simulation process	43
3.4	Unstructured grid for computational domain of solar air heater with inclined fins	45
3.5	Comparison between Nusselt number using different CFD turbulence models with Dittus-Boelter equation for smooth duct of a solar air heater	49
3.6	Convergence for residuals criteria	50
3.7	Grid Independence Test	52
3.8	Comparison of average Nusselt number for the present study results with the results of	53
3.9	Flowchart of the RSM modeling approach for optimal design	55

4.1	Variation of average Nusselt Number with Reynolds number at different value of length $e=1.5,2,2.5$ mm for fixed value of $P=20$ mm and $\alpha=30^\circ$	59
4.2	Variation of average Nusselt Number with Reynolds number at different value of length e = 1.5, 2, 2.5 mm for fixed value of P = 20 mm and α = 45°	59
4.3	Variation of average Nusselt Number with Reynolds number at different value of length e = 1.5, 2, 2.5 mm for fixed value of P = 20 mm and α = 60°	60
4.4	Variation of average Nusselt Number with Reynolds number at different slant angle $\alpha=30^\circ,45^\circ$ and 60° for fixed value of $P=15mm$ and $e=1.5mm$	61
4.5	Variation of average Nusselt Number with Reynolds number at different slant angle $\alpha=30^\circ,45^\circ$ and 60° for fixed value of $P=20$ mm and $e=1.5$ mm	61
4.6	Variation of average Nusselt Number with Reynolds number at different slant angle $\alpha=30^\circ, 45^\circ$ and 60° for fixed value of $P=25$ mm and $e=1.5$ mm	62
4.7	The contour of velocity at Re = 16000 for (a) e = 1.5 mm, P = 20 mm and α = 30° (b) e = 1.5 mm, P = 20 mm and α = 45°, (c) e = 1.5 mm, P = 20 mm and α = 60°	63
4.8	The contour of velocity at Re = 16000 for (a) e = 2.0 mm, P = 20 mm and $\alpha = 30$ ° (b) e = 2.0 mm, P = 20 mm and $\alpha = 45$ °, (c) e = 2.0 mm, P = 20 mm and $\alpha = 60$ °	64
4.9	The contour of velocity at Re = 16000 for (a) e = 2.5 mm, P = 20 mm and α = 30°(b) e = 2.5 mm, P = 20 mm and α = 45°, (c) e = 2.5 mm, P = 20 mm and α = 60°	65
4.10	Variation of average Nusselt Number with Reynolds number at different pitch P = 15 mm, 20 mm and 25 mm for fixed value of α = 30° and e = 1.5 mm	66
4.11	Variation of average Nusselt Number with Reynolds number at different pitch P = 15 mm, 20 mm and 25 mm for fixed value of α = 45° and e = 1.5 mm	67
4.12	Variation of average Nusselt Number with Reynolds number at different pitch P = 15 mm, 20 mm and 25 mm for fixed value of α = 60° and e = 1.5 mm	67

4.13	Variation of friction factor with Reynolds number at different value of length $e = 1.5, 2, 2.5$ mm for fixed value of $P = 20$ mm and $\alpha = 30^{\circ}$	68
4.14	Variation of friction factor with Reynolds number at different value of length $e=1.5,2,2.5$ mm for fixed value of P=20 mm and $\alpha=45^\circ$	69
4.15	Variation of friction factor with Reynolds number at different value of length $e=1.5,2,2.5$ mm for fixed value of $P=20$ mm and $\alpha=60^\circ$	69
4.16	Variation of Friction Factor with Reynolds Number at different slant angle $\alpha=30^\circ,45^\circ$ and 60° for fixed value of P = 15mm and e = 1.5 mm	70
4.17	Variation of Friction Factor with Reynolds Number at different slant angle $\alpha=30^\circ,45^\circ$ and 60° for fixed value of $P=20$ mm and $e=1.5$ mm	71
4.18	Variation of Friction Factor with Reynolds Number at different slant angle $\alpha = 30^{\circ}$, 45° and 60° for fixed value of P = 25 mm and e = 1.5 mm	71
4.19	The contour of pressure at Re = 16000 for (a) e = 1.5 mm, P = 20 mm and $\alpha = 30^{\circ}$ (b) e = 1.5 mm, P = 20 mm and $\alpha = 45^{\circ}$, (c) e = 1.5 mm, P = 20 mm and $\alpha = 60^{\circ}$	73
4.20	The contour of pressure at Re = 16000 for (a) e = 2.0 mm, P = 20 mm and $\alpha = 30^{\circ}$ (b) e = 2.0 mm, P = 20 mm and $\alpha = 45^{\circ}$, (c) e = 2.0 mm, P = 20 mm and $\alpha = 60^{\circ}$	74
4.21	The contour of pressure at Re = 16000 for (a) e = 2.5 mm, P = 20 mm and α = 30°(b) e = 2.5 mm, P = 20 mm and α = 45°, (c) e = 2.5 mm, P = 20 mm and α = 60°	75
4.22	Variation of friction factor with Reynolds number at different value of length $P=15, 20, 25$ mm for fixed value of $e=1.5$ mm and $\alpha=30^\circ$	76
4.23	Variation of friction factor with Reynolds number at different value of length $P=15,20,25$ mm for fixed value of $e=1.5$ mm and $\alpha=45^\circ$	77
4.24	Variation of friction factor with Reynolds number at different value of length $P=15,20,25$ mm for fixed value of $e=1.5$ mm and $\alpha=60^\circ$	77
4.25	Comparison of numerical and predicted values of RSM model for THPP	85
4.26	Normal probability plot residuals for THPP	86
4.27	Effect of Reynolds number and length of fin on THPP (a) 3D surface (b) contour plot	87
4.28	Effect of Reynolds number and pitch of fin on THPP (a) 3D surface (b) contour plot	88

4.29	Effect of Reynolds number and slant angle of fin on THPP (a) 3D surface (b) contour plot	89
4.30	Ramp function graph of desirability for solar air heater duct having inclined fins	91
4.31	Bar graph of desirability for solar air heater duct having inclined fins	91
4.32	Bar graph of Comparison between present study and previous work results	93



LIST OF NOMENCLATURE AND ABBREVIATIONS

C_P	Specific heat of air, J/kg K
D	Equivalent or hydraulic diameter of duct, mm
e	Fin length, mm
h	Heat transfer coefficient, W/m ² K
Н	Depth of duct, mm
I	Turbulent intensity/intensity of solar radiation, $\ensuremath{W/m^2}$
K	Thermal conductivity of air, W/mK
L	Length of duct, mm
<i>L</i> 1	Inlet length of duct, mm
L2	Test length of duct, mm
L3	Outlet length of duct, mm
m	Mass flow rate, kg/s
ΔΡ	Pressure drop, Pa
P	Pitch, mm
U	Velocity of air in the duct, m/s
W	Width of duct, mm
SAH	Solar air heater
CFD	Computational fluid dynamics
RSM	Response surface methodology
CCD	Central composite design
ANOVA	Analysis of variance
$x_1, x_2 \dots x_n$	Independent input variables

Dimensionless parameters

F Friction factor

 f_s Friction factor for smooth surface

Nu Nusselt number

 Nu_s Nusselt number for smooth duct

Pr Prandtl number

Re Reynolds number

THPP Thermo-hydraulic performance parameter

THPPs Thermo-hydraulic performance parameter for smooth duct

W/D Duct aspect ratio

Greek symbols

M Dynamic viscosity, Ns/m²

 μ_t Turbulent viscosity, Ns/m²

P Density of air, kg/m³

A Angle of attack, (°)

K Turbulent kinetic energy, m²/s²

Γ Molecular thermal diffusivity, m²/s

 Γ_t Turbulent thermal diffusivity, m²/s

CHAPTER 1

INTRODUCTION

1.1 Background of the Study

The modern world of today and tomorrow is undergoing an uncertain period with respect to the provision of energy and power as a result of fossil fuel shortages which lead to increased costs and worldwide economic instability. Therefore, it is more vital now than ever to search for alternate means of power generation i.e. sustainable energy in order to fulfill our requirements. As a result, a large number of nations are exploring various technologies centered on using the power of the Sun as the primary source and improving the efficiencies of solar thermal systems (Gawande *et al.*, 2016a).

A particular system used for heating purposes is the solar air heater that works on the basis of solar power. It is utilized in various functions such as crops drying of industrial products as well as for heating in various residential schemes. In this system, heat is transformed from the plate that absorbs to air by the way of convective heat transfer (El-Sebaii *et al.*, 2011a).

Since this system uses the fluid all around us (air) as its transfer material its complexity is much lower than other alternative heating systems and result in simple design and low maintenance requirements (Ammari, 2003). On the contrary, air type solar collectors have two disadvantages i.e. low thermal air capacity and a low absorber to air heat transfer coefficient (Fudholi *et al.*, 2011).

More recently, these solar air heaters operating through forced convection have been the subject of many extensive types of research and scrutiny. Various methodologies have been proposed to improve the operating and economic potentials of the solar air heater which seek the improvement in coefficient of the heat transfer and thus improve the efficiency of the system. It has been shown as a result of these developments that the artificial surface roughness provided on the duct of the solar air heaters, such as fins, ribs, and baffles lead to heat transfer augmentation and superior thermo-hydraulic performance (Lahori *et al.*, 2016).

Since this is still an active research area, the recommended design upgrades of using fins to enhance roughness were introduced in relatively recent times. The fins increase heat transfer area as well as allow for larger turbulence and vortices in the flow which results in improved thermal efficiency.

1.2 Problem Statement

It is a common conversation in today's time that the population of our world is increasing gradually with an alarming rate mainly due to human activities. It is well understood that to do any work everyone needs energy. But, one day the nonrenewable energy resources which give us energy like fossil fuels (coal, gas, oil) will run out with the passage of time. Burning fossil fuels release carbon dioxide when they burn, which not only adds to the greenhouse effect and also increases global warming. As the energy demands are increasing day by day with the increase in population by the next few coming years, it would not be possible to address concerns about such fossil fuels. Nowadays, with this concern, it becomes compulsory to find an alternative resource to meet demands by contributing to the replacement of fossil fuel energy production in order to meet future energy needs and consider the natural environment. Out of all sources of renewable energy, solar energy has many advantages over others (Sachin et al., 2012). It is the cheapest and easily available source having in abundant. This energy can run nearly equal to every application and can be used in any field of life such as; cooking, washing, electricity, in water equipment and on running heavy machinery as well. The best possible way to harness solar energy is to convert it into thermal energy and utilized for desired purposes (Lanjewar et al., 2015). Conventional solar air heater is one of the basic equipment through which solar energy is converted into thermal energy for further usage. Due to the simplicity of the device, they are low-cost and widely used to harness the solar energy available. The development of these devices allows consumers to rely less on conventional energy sources such as fossil fuels for the provision of thermal energy. A highly promising application of solar air heater is in assisting in the drying process of agricultural products and maintaining comfortable internal temperatures of greenhouses and buildings when required. The simplistic structure of the solar air heater also reduces maintenance time and costs. Despite these great advantages, the thermal performance of the solar air heater is found to be poor due to two reasons: the low thermal capacity of air and a low heat transfer coefficient between the absorber plate and air flow through the duct (Kumar et al., 2010). Therefore, it is vital to improving this coefficient of heat transfer so that the effectiveness of the system under consideration is economically justifiable. It has been observed from the literature review that different modifications have been suggested and applied to improve the heat transfer coefficient between the absorber plate and the air and several designs are discussed. However, the importance of having inclined fins on the underside of the absorber plate of solar air heaters has not much been identified and studied. Therefore, there is a need to study the influence of inclined fins arrangement on heat transfer and friction factor in a solar air heater by using a novel CFD study.

1.3 Objectives

The main objective of this research is to increase thermal performance of solar air heater with using inclined fins on the underside of the absorber plate. Towards achieving the main objective, the related aims associated with are identified as follows:

- 1. To propose a new design of solar air heater with inclined fins in order to improve thermal performance.
- 2. To determine the optimum parameters for achieving improved thermal performance in terms of thermo-hydraulic performance parameter (THPP) of the proposed solar air heater by using Design of Experiments (DOE).
- 3. To compare the optimum thermo-hydraulic performance parameter (THPP) results with previous work.

1.4 Scope of the Study

This study covers the generation of a two-dimensional solar air heater model in order to numerically analyze the effect of geometric parameters in order to achieve the maximum THPP. The geometric parameters consist of length of fin (e) which is varied in between values of 1.5mm, 2mm and 2.5mm, slant angle of the inclined fins (α) which is varied between 30°, 45° and 60°, pitch of the fins (p) which is varied in between 15mm, 20mm and 25mm and the Reynolds number which is varied in between the values 4000, 8000, 12,000, 16,000, 20,000 and 24,000. These ranges were selected on the basis of common ranges for the SAH (Rajneesh et al., 2017; Pandey and Bajpai, 2016; Tamna et al., 2014; Yaday and Bhagoria, 2013). The governing equations are formulated considering the fluid flow as a single-phase, incompressible, no radiation heat transfer and fully developed flow (steady-state condition). The solar air heater duct operating on air was used as the working fluid while aluminum was used to develop the absorber plate. The thermo physical characteristics of air as well as the material used for the absorber plate were anticipated to stay constant at a mean bulk temperature. Whilst a 1000 W/m² steady heat flux was sustained on the top wall of the test section to represent the best value of heat flux for solar air heater investigation, and bottom wall is insulated, the inlet section was subjected to a homogenous velocity boundary state and the exit section was subjected to a fixed pressure outlet boundary state (Yadav and Bhagoria, 2013; Yadav & Bhagoria, 2014a; Gawande et al., 2016b).

1.5 Thesis Structure

The research thesis is divided into five main chapters that cover systematically the whole work. Chapter 1 focuses on the research background, problem statement, objectives and scope of the study. Chapter 2 provides a scientific literature review and previous dealing with the issue of enhancing of solar air heater. Chapter 3

presents the methodology adapted, numerical study and software used. Chapter 4 presents in details the analysis results, validation, discussions. Chapter 5 presents the conclusions and recommendation for future work.



REFERENCES

- Agbo, S. N., & Okoroigwe, E. C. (2007). Analysis of thermal losses in the flat-plate collector of a thermosyphon solar water heater. *Research journal of physics*, *I*(1), 35-41.
- Aghaie, A. Z., Rahimi, A. B., & Akbarzadeh, A. (2015). A general optimized geometry of angled ribs for enhancing the thermo-hydraulic behavior of a solar air heater channel—A Taguchi approach. *Renewable Energy*, 83, 47-54.
- Alkilani, M. M., Sopian, K., Alghoul, M. A., Sohif, M., & Ruslan, M. H. (2011). Review of solar air collectors with thermal storage units. *Renewable and Sustainable Energy Reviews*, 15(3), 1476-1490.
- Alkilani, M. M., Sopian, K., & Mat, S. (2011). Fabrication and experimental investigation of PCM capsules integrated in solar air heater. *American Journal of Environmental Sciences*, 7(6), 542.
- Alta, D., Bilgili, E., Ertekin, C., & Yaldiz, O. (2010). Experimental investigation of three different solar air heaters: energy and exergy analyses. *Applied Energy*, 87(10), 2953-2973.
- Ammari, H. D. (2003). A mathematical model of thermal performance of a solar air heater with slats. *Renewable Energy*, 28(10), 1597-1615.
- Bande, Y. M., & Mariah, N. A. (2014). Flat Plate Solar Collectors and Applications: A Review. *Pertanika Journal of Science & Technology*, 22(2).
- Bansal, N. K. (1999). Solar air heater applications in India. *Renewable Energy*, 16(1-4), 618-623.
- Bansal, N. K., & Singh, D. (1985). Analysis of a cylindrical plate matrix solar air heater. *Solar & Wind Technology*, 2(2), 95-100.
- Benli, H., & Durmuş, A. (2009). Performance analysis of a latent heat storage system with phase change material for new designed solar collectors in greenhouse heating. *Solar Energy*, 83(12), 2109-2119.
- Bhagoria, J. L., Saini, J. S., & Solanki, S. C. (2002). Heat transfer coefficient and friction factor correlations for rectangular solar air heater duct having transverse wedge shaped rib roughness on the absorber plate. *Renewable Energy*, 25(3), 341-369.
- Boulemtafes-Boukadoum, A., & Benzaoui, A. (2014). CFD based analysis of heat transfer enhancement in solar air heater provided with transverse rectangular ribs. *Energy Procedia*, *50*, 761-772.
- Budea, S. (2014). Solar air collectors for space heating and ventilation applications—performance and case studies under romanian climatic conditions. *Energies*, 7(6), 3781-3792.

- Chabane, F., Moummi, N., & Benramache, S. (2014). Experimental study of heat transfer and thermal performance with longitudinal fins of solar air heater. *Journal of advanced research*, 5(2), 183-192.
- Chaube, A., Sahoo, P. K., & Solanki, S. C. (2006). Analysis of heat transfer augmentation and flow characteristics due to rib roughness over absorber plate of a solar air heater. *Renewable Energy*, 31(3), 317-331.
- Choudhury, C., Andersen, S. L., & Rekstad, J. (1988). A solar air heater for low temperature applications. *Solar energy*, 40(4), 335-343.
- Ekechukwu, O. V., & Norton, B. (1999). Review of solar-energy drying systems II: an overview of solar drying technology. *Energy conversion and management*, 40(6), 615-655.
- El-Sebaii, A. A., Aboul-Enein, S., Ramadan, M. R. I., Shalaby, S. M., & Moharram, B. M. (2011). Investigation of thermal performance of-double pass-flat and v-corrugated plate solar air heaters. *Energy*, *36*(2), 1076-1086.
- Enibe, S. O. (2002). Performance of a natural circulation solar air heating system with phase change material energy storage. *Renewable Energy*, 27(1), 69-86.
- Evans, B. L., Klein, S. A., & Duffie, J. A. (1985). A design method for active-passive hybrid space heating systems. *Solar energy*, 35(2), 189-197.
- Ferziger, J. H., & Peric, M. (2012). Computational methods for fluid dynamics. Springer Science & Business Media.
- Fudholi, A., Sopian, K., Ruslan, M. H., Othman, M. Y., & Yahya, M. (2011). Analytical and experimental studies on the thermal efficiency of the double-pass solar air collector with finned absorber. *American Journal of Applied Sciences*, 8(7), 716.
- Gao, W., Lin, W., Liu, T., & Xia, C. (2007). Analytical and experimental studies on the thermal performance of cross-corrugated and flat-plate solar air heaters. *Applied Energy*, 84(4), 425-441.
- Garg, H. P., & Hrishikesan, D. S. (1988). Enhancement of solar energy on flat-plate collector by plane booster mirrors. *Solar Energy*, 40(4), 295-307.
- Gawande, V. B., Dhoble, A. S., Zodpe, D. B., & Chamoli, S. (2016a). Experimental and CFD-based thermal performance prediction of solar air heater provided with right-angle triangular rib as artificial roughness. *Journal of the Brazilian Society of Mechanical Sciences and Engineering*, 38(2), 551-579.
- Gawande, V. B., Dhoble, A. S., Zodpe, D. B., & Chamoli, S. (2016b). Experimental and CFD investigation of convection heat transfer in solar air heater with reverse L-shaped ribs. *Solar Energy*, *131*, 275-295.

- Goyal, R. K., & Tiwari, G. N. (1999). Performance of a reverse flat plate absorber cabinet dryer: a new concept. *Energy conversion and Management*, 40(4), 385-392.
- Heidary, H., Abbassi, A., & Kermani, M. J. (2013). Enhanced heat transfer with corrugated flow channel in anode side of direct methanol fuel cells. *Energy conversion and management*, 75, 748-760.
- Hollands, K. G. T., & Shewen, E. C. (1981). Optimization of flow passage geometry for air-heating, plate-type solar collectors. *ASME*, *Transactions*, *Journal of Solar Energy Engineering*, 103, 323-330.
- Karagoz, S. (2015). Investigation of thermal performances of "S-shaped" enhancement elements by response surface methodology. *Heat and Mass Transfer*, 51(2), 251-263.
- Karmare, S. V., & Tikekar, A. N. (2007). Heat transfer and friction factor correlation for artificially roughened duct with metal grit ribs. *International Journal of Heat and Mass Transfer*, 50(21-22), 4342-4351.
- Karmare, S. V., & Tikekar, A. N. (2010). Analysis of fluid flow and heat transfer in a rib grit roughened surface solar air heater using CFD. *Solar Energy*, 84(3), 409-417.
- Karwa, R., Solanki, S. C., & Saini, J. S. (1999). Heat transfer coefficient and friction factor correlations for the transitional flow regime in rib-roughened rectangular ducts. *International Journal of heat and mass transfer*, 42(9), 1597-1615.
- Kaushal, M., Dhiman, P., Singh, S., & Patel, H. (2015). Finite volume and response surface methodology based performance prediction and optimization of a hybrid earth to air tunnel heat exchanger. *Energy and Buildings*, 104, 25-35.
- Kennedy, C. E. (2002). Review of mid-to high-temperature solar selective absorber materials (No. NREL/TP-520-31267). National Renewable Energy Lab., Golden, CO.(US)
- Klevinskis, A., & Vytautas, B. (2011). "Analysis of a Flat-Plate Solar Collector." *Mokslas Lietuvos ateitis* 3: 39–43.
- Kumar, A., Saini, R. P., & Saini, J. S. (2014). A review of thermohydraulic performance of artificially roughened solar air heaters. *Renewable and Sustainable Energy Reviews*, 37, 100-122.
- Kumar, A., Bhagoria, J. L., & Sarviya, R. M. (2009). Heat transfer and friction correlations for artificially roughened solar air heater duct with discrete W-shaped ribs. *Energy Conversion and Management*, 50(8), 2106-2117.
- Kumar, A., Kumar, R., & Behura, A. K. (2017). Investigation for Jet Plate Solar Air Heater with Longitudinal Fins. *International Research Journal of Advanced Engineering and Science*, 2(3): 112–19.

- Kumar, R., & Chand, P. (2017). Performance enhancement of solar air heater using herringbone corrugated fins. *Energy*, 127, 271-279.
- Kumar, R., & Kumar, A. (2017). Computational fluid dynamics based study for analyzing heat transfer and friction factor in semi-circular rib-roughened equilateral triangular duct. *International Journal of Numerical Methods for Heat & Fluid Flow*, 27(4), 941-957.
- Kumar, S., & Saini, R. P. (2009). CFD based performance analysis of a solar air heater duct provided with artificial roughness. *Renewable energy*, 34(5), 1285-1291.
- Kumar, T. S., Thakur, N. S., Kumar, A., & Mittal, V. (2010). Use of artificial roughness to enhance heat transfer in solar air heaters-a review. *Journal of Energy in Southern Africa*, 21(1), 35-51.
- Kumar, R., Kaushik, S. C., & Garg, H. P. (1995). Analytical study of collector solar-gain enhancement by multiple reflectors. *Energy*, 20(6), 511-522.
- Mahesh, M., Sivakumar, S. (2016). "Mathematical Modelling and Analysis of Corrugated Black Coated Solar Flat Plate Collector A Review.": International Conference on Explorations and Innovations in Engineering & Technology 1–6.
- Malhotra, A., Garg, H. P., & Rani, U. (1980). Minimizing convective heat losses in flat plate solar collectors. *Solar energy*, 25(6), 521-526.
- Mashilkar, B., & Varkute, N. (2014). Numerical Investigation of Heat Transfer Augmentation in a Rectangular Solar Air Heater Duct. *Internatinal journal of engineering research and technology*, 3(10): 1245–48.
- Mohammadi, K., & Sabzpooshani, M. (2013). Comprehensive performance evaluation and parametric studies of single pass solar air heater with fins and baffles attached over the absorber plate. *Energy*, *57*, 741-750.
- Momin, A. M. E., Saini, J. S., & Solanki, S. C. (2002). Heat transfer and friction in solar air heater duct with V-shaped rib roughness on absorber plate. *International journal of heat and mass transfer*, 45(16), 3383-3396.
- Moradi, R., Kianifar, A., & Wongwises, S. (2017). Optimization of a solar air heater with phase change materials: Experimental and numerical study. *Experimental Thermal and Fluid Science*, 89, 41-49.
- Lahori, R., Gupta, V., & Yadav, A. (2016). A Review on Different Methods Used for Performance Enhancement of Solar Air Heater. *J. Energy Technol. Policy*, 6(4), 1-7.
- Omojaro, A. P., & Aldabbagh, L. B. Y. (2010). Experimental performance of single and double pass solar air heater with fins and steel wire mesh as absorber. *Applied energy*, 87(12), 3759-3765.

- Omni, (2018). Percentage increase calculator. Retrieved from https://www.omnicalculator.com/math/percentage-increase
- Ozgen, F., Esen, M., & Esen, H. (2009). Experimental investigation of thermal performance of a double-flow solar air heater having aluminium cans. *Renewable Energy*, 34(11), 2391-2398.
- Oztop, H. F., Bayrak, F., & Hepbasli, A. (2013). Energetic and exergetic aspects of solar air heating (solar collector) systems. *Renewable and Sustainable Energy Reviews*, 21, 59-83.
- Pakdaman, M. F., Lashkari, A., Tabrizi, H. B., & Hosseini, R. (2011). Performance evaluation of a natural-convection solar air-heater with a rectangular-finned absorber plate. *Energy conversion and management*, 52(2), 1215-1225.
- Pandey, N. K., & Bajpai, V. K. (2016). Experimental investigation of heat transfer augmentation using multiple arcs with gap on absorber plate of solar air heater. *Solar Energy*, 134, 314-326.
- Patil, A. K., Saini, J. S., & Kumar, K. (2012). Heat transfer and friction characteristics of solar air heater duct roughened by broken V-shape ribs combined with staggered rib piece. *Journal of Renewable and Sustainable Energy*, 4(1), 013115.
- Prasad, B. N., & Saini, J. S. (1988). Effect of artificial roughness on heat transfer and friction factor in a solar air heater. *Solar Energy*, 41(6), 555-560.
- Prasad, S. B., Saini, J. S., & Singh, K. M. (2009). Investigation of heat transfer and friction characteristics of packed bed solar air heater using wire mesh as packing material. *Solar Energy*, 83(5), 773-783.
- Ramadan, M. R. I., El-Sebaii, A. A., Aboul-Enein, S., & El-Bialy, E. (2007). Thermal performance of a packed bed double-pass solar air heater. *Energy*, 32(8), 1524-1535.
- Sahu, J. N., Acharya, J., & Meikap, B. C. (2009). Response surface modeling and optimization of chromium (VI) removal from aqueous solution using Tamarind wood activated carbon in batch process. *Journal of hazardous materials*, 172(2-3), 818-825.
- Saini, S. K., & Saini, R. P. (2008). Development of correlations for Nusselt number and friction factor for solar air heater with roughened duct having arc-shaped wire as artificial roughness. *Solar Energy*, 82(12), 1118-1130.
- Satcunanathan, S., & Deonarine, S. (1973). A two-pass solar air heater. *Solar energy*, 15(1), 41-49.
- Saxena, A., & El-Sebaii, A. A. (2015). A thermodynamic review of solar air heaters. *Renewable and Sustainable Energy Reviews*, 43, 863-890.

- Sethi, M., & Thakur, N. S. (2012). Correlations for solar air heater duct with dimpled shape roughness elements on absorber plate. *Solar Energy*, 86(9), 2852-2861.
- Singh, S., & Dhiman, P. (2014). Thermal and thermohydraulic performance evaluation of a novel type double pass packed bed solar air heater under external recycle using an analytical and RSM (response surface methodology) combined approach. *Energy*, 72, 344-359.
- Singh, S., Chander, S., & Saini, J. S. (2011). Heat transfer and friction factor correlations of solar air heater ducts artificially roughened with discrete V-down ribs. *Energy*, *36*(8), 5053-5064.
- Singh Yadav, A., & Bhagoria, J. L. (2015). Numerical investigation of flow through an artificially roughened solar air heater. *International Journal of Ambient Energy*, 36(2), 87-100.
- Standard, A. S. H. R. A. E. (1977). Methods of testing to determine the thermal performance of collectors. *American Society of Heating, Refrigerating and Air conditioning Engineers Inc.*, Atlanta, GA, 93-77.
- Tamna, S., Skullong, S., Thianpong, C., & Promvonge, P. (2014). Heat transfer behaviors in a solar air heater channel with multiple V-baffle vortex generators. *Solar Energy*, 110, 720-735.
- Thakur, D. S., Khan, M. K., & Pathak, M. (2017). Performance evaluation of solar air heater with novel hyperbolic rib geometry. *Renewable Energy*, 105, 786-797.
- Tu, J., Yeoh, G. H., & Liu, C. (2012). Computational fluid dynamics: a practical approach. Butterworth-Heinemann.
- Tyagi, R. K., Ranjan, R., & Kishore, K. (2014). Performance studies on flat plate solar air heater subjected to various flow patterns. *Applied Solar Energy*, 50(2), 98-102.
- Ucar, A., & Inalli, M. (2006). Thermal and exergy analysis of solar air collectors with passive augmentation techniques. *International communications in heat and mass transfer*, 33(10), 1281-1290.
- Webb, R. L., & Eckert, E. R. G. (1972). Application of rough surfaces to heat exchanger design. *International Journal of Heat and Mass Transfer*, 15(9), 1647-1658.
- Womac, A. R., Tompkins, F. D., & DeBusk, K. E. (1985). Evaluation of solar air heaters for crop drying. *Energy in agriculture*, 4, 147-157.
- Yadav, A. S., & Bhagoria, J. L. (2013). Heat transfer and fluid flow analysis of solar air heater: a review of CFD approach. *Renewable and Sustainable Energy Reviews*, 23, 60-79.

- Yadav, A. S., & Bhagoria, J. L. (2014a). A numerical investigation of square sectioned transverse rib roughened solar air heater. *International Journal of Thermal Sciences*, 79, 111-131.
- Yadav, A. S., & Bhagoria, J. L. (2014b). Heat transfer and fluid flow analysis of an artificially roughened solar air heater: a CFD based investigation. *Frontiers in Energy*, 8(2), 201.
- Yadav, A. S., & Bhagoria, J. L. (2013). A CFD (computational fluid dynamics) based heat transfer and fluid flow analysis of a solar air heater provided with circular transverse wire rib roughness on the absorber plate. *Energy*, 55, 1127-1142.
- Yeh, H. M. (2012). Upward-type flat-plate solar air heaters attached with fins and operated by an internal recycling for improved performance. *Journal of the Taiwan Institute of Chemical Engineers*, 43(2), 235-240.
- Yeh, H. M., & Ho, C. D. (2009). Solar air heaters with external recycle. *Applied Thermal Engineering*, 29(8-9), 1694-1701.