

UNIVERSITI PUTRA MALAYSIA

PARAMETRIC STUDY OF PORTABLE SOLAR-POWERED HEAT EXCHANGER USING COMPUTATIONAL FLUID DYNAMICS

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Thesis Submitted to the School of Graduate Studies, Universiti Putra Malaysia, in Fulfillment of the Requirements for the Degree of Master of Science

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DEDICATION

TO MY BELOVED PARENTS AND MY WIFE NORFARAHAIN WHOSE SUPPORT AND UNDERSTANDING HELPED MAKE THIS POSSIBLE

> SINCERELY TENGKU MUHAMMAD FAHMI

Abstract of thesis presented to the Senate of Universiti Putra Malaysia in Fulfillment of the requirements for the degree of Master of Science

PARAMETRIC STUDY OF PORTABLE SOLAR-POWERED HEAT EXCHANGER USING COMPUTATIONAL FLUID DYNAMICS

By

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Water is important for human. Its importance is highlighted by the United Nation as decade for clean water in the world populations suffer lack of portable water in stressed area such as Africa and South East Asia. Therefore, solar powered water distillation system is important, especially for portable devices seems necessary. This study is concerned with parametric of heat exchanger for a portable telescopic solar powered water distiller. The water supply for the distiller is untreated water from rain water, tube well or flood water. The solar powered water distiller was a Master thesis project completed in 2014 but the heat exchanger requires further improvement on design and fabrication. Presently, the water distiller can produce 10L per day in good sunny weather. The shell and tube heat exchanger with parabolic trough collector (PTC) have potential to increase performance using computational fluid dynamics to produce clean water. Design geometry of heat exchanger was based on the conditions of high-pressure difference between the fluids. The setting of cooling temperature of water inlet is 31°C and water outlet is 29.8°C. However, the tube for fluid inlet is water vapour at 82°C and the temperature outlet is at 38°C. Water vapour comes into tube heat exchanger from the parabolic thermal collector (PTC). The methodology involves using computational fluid dynamics RNG k-epsilon turbulence flow model. The model was applied using CFD (computational fluid dynamics) package conjunction with conjugated heat transfer to predict the flow behavior inside the heat exchanger for both vapour and liquid water flow. Both heat conduction and heat convection are taken into account for this 3D computational method to determine fluid flow and temperature distribution. Parameters considered for design (shell diameter, number of tubes, baffles, temperature difference, velocity of fluid and heat transfer coefficient involved) can be used to develop optimum design of heat exchanger and also through selection of suitable materials. The material construction is a very important aspect in the design for heat exchanger to ensure operating conditions fabrication technique and safe drinking water is produced at a minimum cost and low maintenance. Selected materials have to view the shape design are able to withstand high temperatures and pressures for heat exchanger to perform efficiently. The analysis compares actual

experimental result and defines the suitable material and the best thermal efficiency on the shell in tube heat exchanger. A multiple baffled shell-and-tube heat exchanger design performance is evaluated numerically using CFD (computational fluid dynamics) modeling approach. The field heat exchanger consists of two tubes inside a 0.535 m long and 0.140 m diameter shell. The simulated results are found to be in good agreement with the experiment data. Different materials for the heat exchanger used are stainless steel, copper and aluminum and their performances for a heat exchanger are compared. From the results, it is found that the temperature difference between the vapour outlet and water outlet is least in the case of copper and aluminum as a solid material. Thus, it can be concluded that the copper and aluminum material is giving highest heat transfer to the water. Using copper for heat exchanger can be very expensive and the produced water is not safe to be used for drinking compared to water produced with stainless steel. High heat transferred to the water will help in the sanitising process of the water. Sanitising is a process in which water is cleaned by the use of heat, to turn the water into a usable form.



KAJIAN BERPARAMETER PENUKARAN HABA BAGI SOLAR MUDAH ALIH BERKUASA MENGGUNAKAN PENGIRAAN DINAMIK BENDALIR

Oleh

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Air adalah keperluan asas bagi manusia. Pertubuhan Bangsa-bangsa Bersatu menyatakan bahawa sebahagian besar populasi dunia mengalami kekurangan air bersih mudah alih seperti di Afrika dan Asia Tenggara. Oleh itu, penyulingan air berkuasa solar adalah penting terutamanya bagi alatan mudah alih. Kajian ini adalah mengenai parameter penukaran haba bagi penyuling air berkuasa solar mudah alih. Sumber air yang dibekalkan adalah air yang tidak dirawat yang didapati dari perigi, air hujan atau air banjir. Penyuling air berkuasa solar ini adalah dari projek tesis Master yang disiapkan pada 2014 tetapi penukar haba masih memerlukan peningkatan yang lebih lagi dari segi reka bentuk dan fabrikasi. Penyuling air boleh menghasilkan 10L air sehari ketika cuaca cerah. Kelompong dan tiub penukar haba dengan pengumpul palung parabola (PTC) mampu meningkatkan prestasi dengan menggunakan pengiraan dinamik bendalir untuk menghasilkan air bersih. Reka bentuk geometri penukar haba adalah berdasarkan perbezaan keadaan tekanan tinggi antara cecair. Tetapan suhu penyejukan pada aliran air masuk adalah 31°C dan aliran air keluar adalah 29.8°C. Walau bagaimanapun, wap air yang memasuki tiub masuk adalah pada suhu 82°C manakala suhu keluar adalah 38°C. Wap air memasuki tiub penukar haba dari pengumpul palung parabola (PTC). Metodologi yang terlibat adalah menggunakan pengiraan dinamik bendalir model aliran gelora RNG k-epsilon. Model aliran gelora diaplikasikan menggunakan pakej CFD (pengiraan dinamik bendalir) sealiran dengan penukar haba yang ditasrifkan untuk menjangka sifat aliran di dalam penukar haba bagi kedua-dua aliran wap air dan air. Kedua-dua pengaliran haba dan perolakan haba terlibat di dalam cara pengiraan 3D untuk menentukan aliran cecair dan penyebaran suhu. Parameter yang diambil kira untuk reka bentuk (iaitu, diameter kelompong, bilangan tiub, sesekat, perbezaan suhu, halaju cecair dan pekali penukaran haba yang terlibat) boleh digunakan untuk mereka bentuk penukar haba yang berprestasi maksimum disamping membuat pemilihan bahan-bahan yang sesuai. Pemilihan bahan adalah aspek yang sangat penting dalam reka bentuk penukar haba untuk memastikan teknik keadaan fabrikasi yang beroperasi dan air minuman yang selamat dihasilkan pada kos yang minimum dan penyelenggaraan yang rendah. Bahan yang dipilih mestilah sesuai dengan reka bentuk dan boleh menahan suhu dan tekanan yang tinggi dalam menjadikan penukar haba berfungsi dengan cekap. Analisis dilakukan untuk membandingkan keputusan eksperimen yang seterusnya menentukan bahan yang sesuai pada kelompong di tiub penukar haba. Beberapa hasil reka bentuk penukar haba dengan kelompong-dan-tiub bersesekat dinilai secara berangka menggunakan pendekatan model CFD (pengiraan dinamik bendalir). Ruang penukar haba terdiri daripada dua tiub kelompong yang berukuran 0.535 m panjang dan berdiameter 0.140 m. Dapatan kajian adalah menyokong data dari eksperimen. Bahanbahan yang berbeza seperti besi tahan karat, tembaga dan aluminium digunakan pada penukar haba dan hasilnya diukur. Didapati bahawa penukar haba yang diperbuat dari tembaga dan aluminium memberikan perbezaan suhu paling sedikit di antara saluran keluar wap dan saluran keluar air. Oleh itu, ia dapat disimpulkan bahawa tembaga dan aluminium memberikan penukaran haba paling tinggi ke atas air. Pengunaan tembaga pada penukar haba memerlukan kos yang tinggi dan air yang dihasilkan tidak selamat untuk dimin<mark>um berbanding dengan pe</mark>nggunaan besi tahan karat. Haba tinggi yang dipindahkan kepada air akan membantu pemprosesan air. Pembersihan adalah proses di mana air dibersihkan dari mikroorganisma dengan penggunaan haba untuk menjadikan air selamat digunakan.

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LIST OF NOMENCLATURES

A	Total surface area (m²)
A_c	Inner surface area of the tube under cleaned condition (m ²)
A_{f}	Inner surface area of the tube under fouling condition (m ²)
	Inner tube surface (m ²)
A_{i}	Outer tube surface (m²)
Ao	
A_s	Fictitious cross sectional area (m²)
В	Baffle space (m)
C	Clearance between tubes (constant numbers)
C	Fluid specific heat (W/K)
C_{p}	Specific heat at constant pressure (J/kg.K)
D	Tube diameter (m)
D_e	Equivalent diameter (m) for square pitch layout
D_{i}	Inner tube diameter (m)
D_{ns}	Minimum inside shell nozzle diameter (m)
D_{o}	Outer tube diameter (m)
\underline{D}_{s}	Shell diameter (m)
F	LMTD correction factor
f	Friction factor
G_s	Gravitational acceleration (m/s²)
	Mass velocity in shell side (kg/m².s)
h	Heat transfer coefficient (W/m².K)
h _m	Modified heat transfer coefficient (W/m².K)
i _{fg}	Enthalpy (J/kg)
k	Thermal conductivity (W/m.K)
k_l	Liquid phase thermal conductivity (W/m.K)
L	Length of heat exchanger tube (m)
L_{V}	Vapour latent heat (kJ/kg)
m	Fluid mass (kg)
m	Fluid mass flow rate (kg/sec)
N _t	Number of tube
$N_{\rm u}$	Nusselt number
P_{t}	Tube pitch (m)
Q	Heat transfer (W)
q	Heat flow (kJ/sec)
R_{bs}	The ratio of baffle space to shell diameter $(\frac{B}{D_a})$
D	Fouling resistance factor (m ² /K _• W)
R_f	Tube radius (mm)
r Re	Reynolds number
	Total thermal resistance (m²/W•K)
R_t	Conduction resistance
R _w T	Temperature (K)
	Temperature (K) Temperature of fluid A on wall side one (K)
T_A	Free stream condition temperature (°C or K)
T_{∞}	Temperature of fluid B on wall side tow (K)
T _B	Base surface temperature (°C or K)
T_{b}	Dase surface temperature (C or K)

t Wall thickness (m) U Overall heat transfer coefficient (W/m².K) Fluid velocity in tube side (m/s) u_{m} Overall heat transfer coefficient for clean surface (W/m².K) u_c Overall heat transfer coefficient for fouling surface u_f $(W/m^2.K)$ V_{ns} Maximum fluid velocity in shell side Vapor quality X ΔΡ Pressure drop (Pa) ΔΤ Local temperature difference between two fluids (K) ΔT_{lm} logarithm-mean temperature difference (k) Suitable mean temperature difference across the heat $\Delta T_{\rm m}$ exchanger (K)

Temperature different between two sides wall (K)

Greek symbols

 $\Delta T_{overall}$

Δx

ε $η_0$ μ Overall surface efficiency
 Dynamic viscosity (kg/m.s) ρ Fluid density (kg/m³)
 Coefficient
 Void fraction of the vapour

Wall thickness (m)

Subscripts

c Cold fluid
f Fouling
g Gas phase
h Hot fluid
i Inner, Inside
Liquid phase
m Mean

max Maximum Minimum

Outer, Outside, Overall

Shell side

t Tube, Thermal, Total

v Vapor phase 1 Inlet, Side 1 2 Outlet, Side 2

LIST OF ABBREVATIONS

ASS Active Solar Still

CL Tube Layout Constant

CPC Compound Parabolic Collector
CFD Computational Fluid Dynamics
CTC Cylindrical Trough Collector

CSS Conventional Solar Still

CSSP Continuous Single Shell-Pass

CSP Concentrated Solar Power

CH Continuous Helical

DCMD Direct Contact Membrane Distillation

DNS Direct Numerical Simulation

EHT Enhancing Heat Transfer

HBHE Helical Baffle Heat Exchanger

HE Heat Exchanger

HFC Heliostat Filed Collector
HTC Heat Transfer Coefficient

FPD Fuzzy Proportional Derivative

FVM Finite Volume Method

ID Inside Diameter

ICS Integrated Collector Storage

IEHX Indirect Evaporative Heat Exchanger

LES Large Eddy Simulation
LFR Linear Fresnel reflector

LMTD Log-Mean Temperature Difference

MCHX Microchannel Heat Exchanger

NTU Number of Transfer Unit

OD Outside Diameter

OECD Organization for Economic Cooperation And

Development

PCM Phase Change Material

PDE Partial Differential Equations

PDR Parabolic Dish Reflector

PR Pitch Ratio

PTC Parabolic Trough Collector

RANS Reynolds-Averaged Navier Stoke

SSP Shallow Solar Pond

SST Shear Stress Transport

SRS Suns River Still

STHX Shell-and-Tube Heat Exchanger

SWH Solar Water Heaters

TEAM Tubular Exchanger Manufacturers Association

TOT Twisted Oval Tube

TSS Tubular Solar Still

UPM Universiti Putra Malaysia

UDF Under-Dynamic Factor

URF Under-Relaxation Factor

URANS Unsteady Reynolds-Averaged Navier Stoke

WHO World Health Organization

CHAPTER 1

INTRODUCTION

1.1 Background

In the world today, humankind life depends on basic needs that include resources (oxygen, water, food and lodging). Many countries including both developed and under-developed countries are struggle with drinking water supply. This problem is rather more severe in the region where flood occurs frequently (seasonal and non-seasonal) and the rise of local poverty issue. In many years, it is believed that solar distiller can solve this problem with the technology advancement and for this purpose, a heat exchanger can be used to increase water distillation.

Solar distillation system based on Figure 1.1 is designed to transform undrinkable water into drinking water utilizing solar energy. To better engineer such design, heat exchanger inside solar distillation system is designed to take into account for its efficiency and productivity. This heat exchanger is based on shell-and-tube type as shown in Figure 1.2. The shell and tube design is greatly considered to have impact on the heat transfer performance, some key parameters are identified the number and size of baffles, number and size of internal tubes. In addition, material selection for the solid shell-and-tube heat exchanger will also have impact on the effectiveness and productivity performance. The construction for heat exchanger prototype for this kind is expensive in cost and time consuming. Therefore, a systematical approach using state-of-the-art computational fluid dynamics (CFD) method is proposed to study qualitatively and quantitatively the performance of the design.

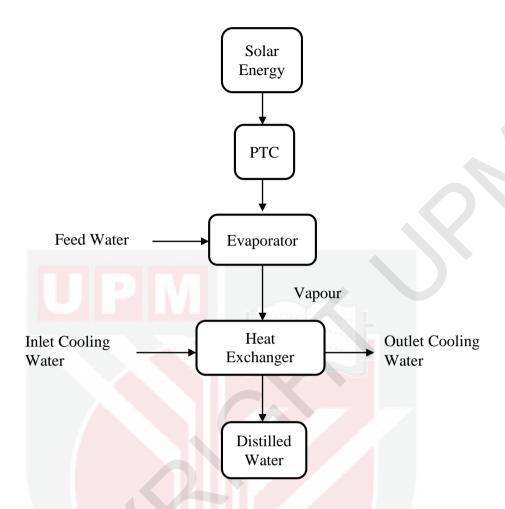


Figure 1.1: Solar powered water distiller system

The simulation results provide further insight into the significance of these design parameters to the overall efficiency of heat transfer performance. The numerical approach can also analyze in details of the fluid flow field, which is difficult to be measured or visualized from experimental work done to many parameters such as vapor mixture, heat transfer rate and water prediction. Using CFD analysis approach, many present problems encountered in experiments can be overcome without trial and error with expensive physical prototype and with much short turn-around time. Furthermore, the deployment of CFD process established from this current research on heat exchanger design can potentially maximize the productivity and quality of fresh drinking water that one can achieve based on an optimal design identification.

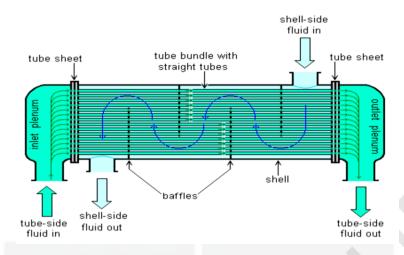


Figure 1.2: Shell-and-tube heat exchanger by Padleckas (2006)

Some literatures including report from Rajappan and Azhaguvelu (2013), stated that to increase the cooling efficiency of normal shell and tube heat exchanger such as parameters of surface temperature, temperature difference, velocity and pressure are crucial. Heat exchanger performance and efficiency are measured through the amount of heat transfer using least area of pressure drop and heat transfer. Overall heat transfer coefficient can be calculated to improve the efficiency of heat exchanger. Power requirement and capital cost of heat exchanger provide the actual area required for this work.

In addition, Stonebraker et al. (2014) have discussed a parabolic solar water distillation system concept performance based on their final assembly set-up. They have identified the key components needed in the design to produce clean drinkable water. Their design consists of 32 degree tilted acrylic film tape trough, absorber tube, sprayed nozzle, distiller with heat exchanger, integrated piping with insulation, Bachmann mini water pump, and distilled water reservoir. Their main objective of this design is to heat water to vaporization. Through their design, condensation rate of 0.5 gph is also measured with optical efficiency at 0.71 and heat radiation of 338.5 W/m².

1.2 Problem Statement

At Universiti Putra Malaysia (UPM), Khiabani (2011) has designed a point-focus solar collector for his system. His shell and tube heat exchanger designed to collect heat by absorbing sunlight using point-focus concentrators, (center receiver system). His shell and tube heat exchanger was designed and simulated using an implicit numerical scheme. Simulation result showed that accumulated mass water greatly depended on the inlet vapor temperature and volume, heat exchanger material, coolant water temperature and coolant volume. He reported that these inexpensive shell and tube heat exchanger could produce 10 L/day. If inlet pressure increase, vapor temperature will decline and thereupon, heat exchanger efficiency tangibility will increase. In general, solar powered water distiller is of fixed type.

Also at UPM, Ali Driver (2014) developed a solar powered water distiller using parabolic trough collector (PTC). He developed line focus concentrators PTC and linear fresnel collectors capable of producing 10L/day on a sunny day. Design factors include focus length, size of PTC, length of heat exchanger. Shell and tube heat exchanger is the most suitable type for solar powered distiller system. Design parameter investigated include shell diameter number of internal tube and temperature of water vapor develop as temperature input for heat exchanger. Recently, the project regarding to design heat exchanger solar powered water distiller using shell-and-tube heat exchanger with 10L/day output and in prediction in 15L/day capacity. The purpose of heat exchanger is to liquefy water vapor generated from the PTC and entering at 82°C into the heat exchanger. The inlet fluid temperature of water liquid is 25°C-28°C and outlet temperature is 35°C. However, water vapor entering the heat exchanger is at 80°C and the distillate water is produced at 30°C. The purpose of looking for potential suitable material is to ensure manufacture ability of that material but still produce high volume of clean water by ensuring a proper heat transfer. The water distiller work involves using CFD for design and analysis of shell and tube heat exchanger by changing the materials used for predicting the heat exchanger to find the temperature difference and heat transfer involved to develop an optimum design of heat exchanger by choosing suitable material and design. ANSYS FLUENT 14 is used to simulate and compare the results with different selected material for the temperature difference between inlet and outlet of heat exchanger.

1.3 Objectives

The main objectives of this research are of the following:

- To predict the performance of outlet temperature for vapor side a shell-andtube heat exchanger for solar powered distiller application.
- To study material selection impact on heat transfer performance of the heat exchanger design.

1.4 Scope and Limitations

This work is based on numerical analysis. The major scope in this project is to find the best suitable material which can be used to manufacture the heat exchanger, the criteria for the best material is to get maximum heat transfer and ability to produce water and also easy to manufacture the heat exchanger using that material. The steps involved in CFD process includes geometry construction, meshing, flow boundary conditions setting, material selection, model simulation, and post-processing. Scope methodology following Haran (2013) as the parameter by experimental and numerical analysis using RNG k-E in steady state condition and boundary conditions materials is based on Ali Driver (2014) and Khaibani (2011). Limitation of selected materials are stainless steel, copper and aluminum as command fabrication. Discussions of the numerical results will also be included such as validation with experimental data, comparing the heat transfer performance of heat exchanger with different materials that are stainless steel, copper, and aluminum. To get an optimum design a

comparative analysis of flow parameters such as fluid flow/solid temperature, pressure and velocity is done. Reynolds number for heat exchanger in laminar around 1866.80, which is the CFD analysis not using transient condition.

The project is limited to the change in the material used for the heat exchanger, no change in velocity, fluid and temperature is done. It is worthwhile to mention that the last activity of prototype built and testing are not required for the current study. This is outside the work scope of the current thesis but it is part of the entire research activity to be considered as complete process.

1.5 Thesis Layout

This thesis is laid out with total 5 chapters based on the following flow format:

It starts with the introduction chapter (Chapter 1) which presents the background of the thesis, and then followed by problem statement, research objectives, work scope and limitations, thesis contribution. Chapter 2 consists of literature review of relevant publications that were reported and studied in the past. These include different solar powered water distiller (background and mechanism), different heat exchanger types and flow arrangements, calculation of overall heat transfer coefficient, selected material properties for heat exchanger calculation using Log Mean Temperature Difference (LMTD) and the Effective-NTU method, satisfactory design, and heat exchanger design considerations.

Chapter 3 documents the methodologies conducted based on the current research. The methodology is broken down in the following different sections: Design methodology based on morphology chart and flow chart, CFD mathematical modeling approach, experiment and simulation validation results, CFD analysis discussion including FLUENT procedure, 3D geometry creation, computational mesh generation, and CFD solution and steps demonstration. Chapter 4 discusses in details of the numerical predictions from current research. This chapter reports the performance of 3 different CFD models comparison, baseline validation against experimental data, and then followed by CFD post-processing technique including contour plots, profile, mass and heat flux, pressure drop and heat transfer. Chapter 5 concludes the findings with remark and suggestion. This chapter also discuss some recommendations for future work.

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LIST OF PUBLICATIONS

- T.M.F.T. Ibrahim, N.M. Adam and C.N. Jaafar, "Parametric Study of Portable Solar-Powered Heat Exchanger Using Computational Fluid Dynamics" Applied Thermal Engineering, EVISE, 03 February 2017 [Submitted]
- Nor Mariah Adam, Mohd Zainal Abidin Abdul Kadir, Tengku Muhammad Fahmi Tengku Ibrahim "A Solar Powered Water Distiller" Intellectual Property Corporation of Malaysia, UI 2015704631, UPM, 13 April 2016
- A.R. Zahari, S.Z.M. Daud, N.A. Zakaria, T.M.F.T. Ibrahim and C. Gomes "Potential of Wind Energy in Empowering Small Islands" Energy for Sustainable Development, Elsevier, 18 July 2016 [Submitted]
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- T.M.F.T. Ibrahim, N.M. Adam and C.N. Jaafar, "Design of Heat Exchanger for Portable Solar Powered Water Distiller Using Computational Fluid Dynamics" Renewable Energy, Elsevier, 23 March 2016 [Submitted]
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