



UNIVERSITI PUTRA MALAYSIA

***NUMERICAL SIMULATION OF THE EFFECT OF CH₄, H₂ AND DIESEL
FUEL MIXTURE ON FOUR STROKE ENGINE***

HAYDER ABDULLAH LUAIBI ALRAZEN

FK 2015 28



**NUMERICAL SIMULATION OF THE EFFECT OF CH₄, H₂ AND DIESEL FUEL
MIXTURE ON FOUR STROKE ENGINE**

By

HAYDER ABDULLAH LUAIBI ALRAZEN

**Thesis Submitted to School of Graduate Studies, Universiti Putra Malaysia, in
Fulfilment of the Requirements for the Degree of Master of Science**

November 2015

COPYRIGHT

All material contained within the thesis, including without limitation text, logos, icons, photographs, and all other artwork, is copyright material of Universiti Putra Malaysia unless otherwise stated. Use may be made of any material contained within the thesis for non-commercial purposes from the copyright holder. Commercial use of material may only be made with the express, prior, written permission of Universiti Putra Malaysia.

Copyright© Universiti Putra Malaysia

Abstract of thesis presented to the Senate of Universiti Putra Malaysia in fulfillment of the
Degree of Master of Science

NUMERICAL SIMULATION OF THE EFFECT OF CH₄, H₂ AND DIESEL FUEL MIXTURE ON FOUR STROKE ENGINE

By

HAYDER ABDULLAH LUAIBI ALRAZEN

November 2015

Chairman : Associate Professor Kamarul Arifin Ahmad, PhD, Ir
Faculty : Engineering

Gaseous fuels have been investigated to be a helpful substitute in compression-ignition engine by researchers. There was extension in the ignition delay of diesel-CH₄ dual-fuel mode as compared with usual diesel fuel mode. Methane has a low flame propagation speed as well as slight flammability whereas hydrogen has the extreme opposite characteristics. As such adding hydrogen can enhance methane's combustion process making it extra convenient in diesel engine application. H₂-Diesel produced many of the unwanted effects such as rapid burning rate and increased diffusivity and reduced ignition energy of hydrogen that may lead to knocking, an impact that is harmful to engine's mechanical durability as well as safety. Methane addition has the ability to make hydrogen combustion stable and smoother which can prevent imperfect combustion. Methane can also lower the combustion temperature of hydrogen so as to repress NO_x emission. In the present study, the author proposes that by adding hydrogen into methane and diesel, it can improve the combustion process. The usage of GAMBIT software was chosen to create the entire computational domain of the engine and for Computational Fluid Dynamics (CFD) the FLUENT code was used. The engine was operated under dual-fuel and tri-fuel modes with different values of excess air (λ) including 1.2, 1.4, 1.6, 1.8, 2, 2.2 and 2.4. Moreover, torque (20.18 N.M), intake temperature (330 K), and engine speed (2000 rpm) were taken constantly at an atmospheric pressure. Diesel-CH₄, diesel-H₂ dual-fuel operation, and diesel-CH₄-H₂ tri-fuel operation were employed in this work. H30-M70, H50-M50 and H70-M30 were designed for the mixtures percent of hydrogen to methane which are 30:70, 50:50 and 70:30 %, respectively, and then used them in the simulations. Due to knocking, the maximum quantity of substitution by hydrogen was limited to 50%. Therefore, the quantity of diesel was employed 50 percent by mass from the total fuel at diesel mode and the other 50 percent was substituted by the methane and hydrogen as mentioned above.

The addition of gaseous fuels increases the peak in-cylinder pressure and peak temperature at both the low and medium values of the exceed air. Meanwhile, at high value of exceeds air, no effects on the peak temperature were noted between Diesel-H70-M30 for tri mode and Diesel-H₂ for dual mode. Compared with CH₄-Diesel at 2.4 exceed air, the peak pressure increases by 28.57% and 33.414% by way of adding the limit value of hydrogen to methane, such as H30-M70 and H50-M50, respectively. Compared with H50-M50, it begins to decrease by 0.726% and 3.81% with H70-M30 and H₂-Diesel operations, respectively, that may be because of the low value of fuels in air compared with other cases. The addition of methane in hydrogen produces a smoother combustion of hydrogen and ascertains that the engine is safe and it has mechanical durability.

Tri-fuel and dual-fuel modes have a similar suppression effect on CO_2 emission but with hydrogen there is more reduction in CO_2 emission compared with methane. However, Diesel- H_2 - CH_4 operations decrease the CO emission compared with the Diesel- CH_4 operation and decrease the NO emission compared with the Diesel- H_2 operation at every exceed air. High hydrogen fraction in methane (H70-M30) is suggested at all exceeds air in order to reduce CO/ CO_2 emissions, whereas low hydrogen fraction in methane (H30-M70) can suppress the uncontrolled hydrogen combustion and limit the increment of the NO emission.

Abstrak tesis yang dikemukakan kepada Senate Universiti Putra Malaysia sebagai memenuhi Sebahagian keperluan untuk Ijazah Master Sains

SIMULASI BERANGKA DARIPADA CH₄, H₂ DAN DIESEL TERHADAP PENGGUNAAN ENJIN 4 STROK

Oleh

HAYDER ABDULLAH LUAIBI ALRAZEN

November 2015

Pengerusi : Professor Madya Kamarul Arifin Ahmad, PhD, PE
Fakulti : Kejuruteraan

Bahan api gas telah ditemui sebagai pengganti berguna dalam enjin mampatan penyalaan oleh penyelidik. Terdapat lanjutan dalam tempoh lengah operasi dwi-bahan api diesel-CH₄ berbanding dengan operasi bahan api diesel biasa. Metana mempunyai kelajuan perambatan api yang rendah serta kemudahbakaran sedikit manakala hidrogen mempunyai ciri-ciri yang bertentangan. Penambahan hidrogen, boleh meningkatkan proses pembakaran metana dan menjadikannya lebih mudah dalam aplikasi enjin diesel. H₂-Diesel menghasilkan pelbagai kesan yang tidak diingini seperti kadar pembakaran yang cepat, peningkatan keterasapan dan pengurangan tenaga pencucuhan hidrogen yang boleh membawa kepada ketukan enjin, kesan yang memudaratkan ketahanan mekanikal enjin dan juga keselamatan. Penambahan metana mempunyai keupayaan untuk menghasilkan pembakaran hidrogen yang stabil dan lancar yang boleh mengelakkan pembakaran tidak normal. Metana juga boleh menurunkan suhu pembakaran hidrogen untuk menindas pelepasan NO_x. Dalam kajian ini, Gambit digunakan untuk mencipta domain pengkomputeran keseluruhan enjin dan komersial Pengkomputeran Dinamik Bendalir (CFD) kod FLUENT digunakan. Enjin ini telah dikendalikan di bawah dwi-bahan api dan mod tri-bahan api dengan perbezaan nilai lebihan udara (λ) termasuk 1.2, 1.4, 1.6, 1.8, 2, 2.2 dan 2.4. Selain itu, daya kilas (20.18 NM), suhu pengambilan (330 K), dan kelajuan enjin (2000) telah ditetapkan pada tekanan atmosfera. Diesel-CH₄, operasi dwi-bahan api diesel-H₂, dan operasi tri-bahan api diesel-CH₄-H₂ telah digunakan dalam penyelidikan ini. Tiga campuran hidrogen-metana daripada 30:70, 50:50 dan 70:30 % hidrogen kepada metana, ditetapkan sebagai H30-M70, M50-H50 dan H70-M30, masing-masing, telah dibeli dan digunakan dalam simulasi ini. Oleh disebabkan pengetukan, jumlah maksima penggantian hidrogen adalah terhad kepada 50%. Oleh itu, kuantiti diesel telah bekerja 50 peratus dengan kadar aliran jisim daripada jumlah bahan api pada mod diesel dan 50 peratus lagi telah dibahagikan terhadap metana dan hidrogen seperti yang dinyatakan di atas.

Kajian mendapati bahawa nilai tekanan puncak dan suhu puncak di dalam silinder telah meningkat dengan penambahan bahan api gas pada nilai exceeds air yang rendah dan sederhana. Perbandingan diantara menggunakan Diesel-H70-M30 untuk mod tri dan Diesel-H₂ untuk mod dual menunjukkan tiada kesan kepada nilai suhu puncak pada nilai exceed air yang tinggi. Semasa exceeds air bernilai 2.4, tekanan puncak meningkat dengan penambahan had hydrogen kepada metana, seperti H30-M70 dan M50-H50 dan mula berkurangan dengan H70-M30 dan operasi H₂-Diesel. Operasi Diesel-H₂-CH₄ mengurangkan pelepasan CO/CO₂ berbanding dengan operasi Diesel-CH₄. Operasi Diesel-

H₂-CH₄ juga mengurangi pelepasan NO berbanding dengan operasi Diesel-H₂ pada setiap exceeds air. Kajian telah mencadangkan bahawa pengurangan pelepasan CO/CO₂ berlaku apabila kandungan bahagian hydrogen di dalam metana adalah tinggi (H70-M30) pada semua keadaan exceeds air. Kandungan bahagian hidrogen yang rendah di dalam metana (H30-M70) boleh menyekat pembakaran hidrogen yang tidak terkawal dan mengehadkan kenaikan pelepasan NO.

ACKNOWLEDGEMENTS

Thanksgiving and praise be to God Almighty for all the blessing, such as health and wellness, to complete this work as well as gave me the knowledge to accomplish it.

Dedicate sincere thanks and appreciation to my supervisor (Assistant Professor Kamarul Arifin Bin Ahmad) where his advices and his comments were tremendous to advance this work and then completing. Also, I give many thanks to my co-supervisor (Assistant Professor Abdul Rahim Abu Talib), where my studies were not abandoned from his important comments which have increased in the research hardness.

Finally, I should not forget my dear wife who supported me by her wide heart and her pretty patience, as well as my father and brothers and sisters who have supported me to complete this thesis.

I certify that a Thesis Examination Committee has met on 19 November 2015 to conduct the final examination of Hayder Abdullah Luaibi Alrazen on his thesis entitled "Numerical Simulation of the Effect of CH₄, H₂ and Diesel Fuel Mixture on Four Stroke Engine" in accordance with the Universities and University Colleges Act 1971 and the Constitution of the Universiti Putra Malaysia [P.U.(A) 106] 15 March 1998. The Committee recommends that the student be awarded the Master of Science.

Members of the Thesis Examination Committee were as follows:

Mohamed Thariq bin Hameed Sultan, PhD

Senior Lecturer
Faculty of Engineering
Universiti Putra Malaysia
(Chairman)

Azmin Shakrine bin Mohd Rafie, PhD

Associate Professor
Faculty of Engineering
Universiti Putra Malaysia
(Internal Examiner)

Faizal bin Mustapha, PhD

Associate Professor Ir.
Faculty of Engineering
Universiti Putra Malaysia
(Internal Examiner)

Mohd Zulkifly Abdullah, PhD

Professor
Universiti Sains Malaysia
Malaysia
(External Examiner)



ZULKARNAIN ZAINAL, PhD

Professor and Deputy Dean
School of Graduate Studies
Universiti Putra Malaysia

Date: 15 December 2015

This thesis was submitted to the Senate of Universiti Putra Malaysia and has been accepted as fulfilment on the requirement for the degree of Master of Science. The members of the supervisory committee were as follows:

Kamarul Arifin Ahmad, PhD

Associate Professor, Ir
Faculty of Engineering
Universiti Putra Malaysia
(Chairman)

Abdul Rahim Abu Talib, PhD

Associate Professor
Faculty of Engineering
Universiti Putra Malaysia
(Member)

BUJANG BIN KIM HUAT, PhD

Professor and Dean
School of Graduate Studies
Universiti Putra Malaysia

Date:

Declaration by graduate student

I hereby confirm that:

- x this thesis is my original work
- x quotations, illustrations and citations have been duly referenced
- x the thesis has not been submitted previously or concurrently for any other degree at any institutions
- x intellectual property from the thesis and copyright of thesis are fully-owned by Universiti Putra Malaysia, as according to the Universiti Putra Malaysia (Research) Rules 2012;
- x written permission must be owned from supervisor and deputy vice –chancellor (Research and innovation) before thesis is published (in the form of written, printed or in electronic form) including books, journals, modules, proceedings, popular writings, seminar papers, manuscripts, posters, reports, lecture notes, learning modules or any other materials as stated in the Universiti Putra Malaysia (Research) Rules 2012;
- x there is no plagiarism or data falsification/fabrication in the thesis, and scholarly integrity is upheld as according to the Universiti Putra Malaysia (Graduate Studies) Rules 2003 (Revision 2012-2013) and the Universiti Putra Malaysia (Research) Rules 2012. The thesis has undergone plagiarism detection software

Signature: _____

Date: _____

Name and Matric No: Hayder Abdullah Luaibi Alrazen GS 38374

Declaration by Members of Supervisory Committee

This is to confirm that:

- x the research conducted and the writing of this thesis was under our supervision;
- x supervision responsibilities as stated in the Universiti Putra Malaysia (Graduate Studies) Rules 2003 (Revision 2012-2013) were adhered to.

Signature: _____

Name of
Chairman of
Supervisory

Committee: Kamarul Arifin Ahmad, PhD

Signature: _____

Name of
Member of
Supervisory

Committee: Abdul Rahim Abu Talib, PhD

TABLE OF CONTENTS

	ABSTRACT	Page
	ABSTRAK	i
	ACKNOWLEDGEMENT	iii
	APPROVAL	v
	DECLARATION	vi
	LIST OF TABLES	viii
	LIST OF FIGURES	xii
	LIST OF ABBREVIATIONS	xiv
		xvii
 	CHAPTER	
1	INTRODUCTION	1
1.1	Background	1
1.2	Problem Statement	3
1.3	Hypothesis	3
1.4	Research Objectives	4
1.5	Scope of Research	4
1.6	Thesis Layout	4
2	LITERATURE REVIEW	6
2.1	Introduction	6
2.2	Alternative Fuels	6
2.2.1	Natural Gas	6
2.2.2	Hydrogen	8
2.2.3	Hydrogen- Hydrocarbon Mixtures	9
2.3	Effect of Hydrogen Addition on Performance	9
2.3.1	Power Output and Thermal Efficiency	9
2.3.2	Duration of Combustion	12
2.3.3	In Cylinder Pressure	14
2.3.4	Heat Release	17
2.3.5	Brake Mean Effective Pressure	20
2.3.6	Specific Energy Consumption	22
2.4	Effect of Hydrogen Addition on Emissions	24
2.4.1	Unburned Hydrocarbon, Carbon Monoxide, and Carbon Dioxide	24
2.4.2	Particulate Matter (PM) & Smoke	26
2.4.3	NO _x Emission	28
2.5	Effect of EGR Rate with Hydrogen Addition	32
2.6	Numerical Method	34
2.6.1	Equations of Motion	34
2.6.2	Turbulent Model	35
2.6.3	Fluent Solver	35
2.6.4	Discretization	36
2.6.5	Pressure Interpolation Schemes	37
2.6.6	Pressure-Velocity Coupling	37
2.6.7	Under-Relaxation Factors	37
2.7	Phenomena Simulated	37
2.7.1	Spray Modeling	37
2.7.2	Ignition Delay (Autoignition) Modeling	38
2.7.3	NO _x Modeling	40
2.7.4	Chemical Reactions Modeling	41

2.8	Summary	43
3	METHODOLOGY	45
3.1	Introduction	45
3.2	Data and Initial Conditions	46
3.3	Computation Tools	48
3.4	Design of Research Flow Chart	49
3.5	Grid Generation	50
3.5.1	Initial Grid Generation	50
3.5.2	Moving Dynamic Mesh	52
3.6	Boundary Conditions and Fluid Properties	55
3.7	Turbulent Model	56
3.8	Fluent Solver	56
3.9	Pressure Interpolation Schemes	56
3.10	Pressure-Velocity Coupling	56
3.11	Iteration Residual and Time Steps	56
3.12	Phenomena Simulated	57
3.12.1	Spray Modeling	57
3.12.2	Chemical Reactions Modeling	58
4	RESULTS AND DISCUSSION	60
4.1	Introduction	60
4.2	Mesh Independent Test	60
4.3	Validation Model	61
4.4	Results and Discussions	62
4.4.1	Combustion Characteristics	63
4.4.2	Emissions	78
4.5	Summary	105
5	CONCLUSIONS AND RECOMMENDATIONS	106
5.1	Recommendation for Future Research	107
	REFERENCES	108
	APPENDICES	118
	BIODATA OF STUDENT	128
	LIST OF PUBLICATIONS	129

LIST OF TABLES

Table	Page
2.1 Average natural gas composition in different countries	8
3.1 Test Cases	48
3.2. Specification of diesel fuel	55
4.1 Definition of percent for methane and hydrogen	63
4.2-a. The development of average temperature under different ratio of gaseous addition and 1.2 exceed air	71
4.2-b. The development of average temperature under different ratio of gaseous addition and 1.4 exceed air	72
4.2-c. The development of average temperature under different ratio of gaseous addition and 1.6 exceed air	73
4.2-d. The development of average temperature under different ratio of gaseous addition and 1.8 exceed air	74
4.2-e. The development of average temperature under different ratio of gaseous addition and 2 exceed air	75
4.2-f. The development of average temperature under different ratio of gaseous addition and 2.2 exceed air	76
4.2-g. The development of average temperature under different ratio of gaseous addition and 2.4 exceed air	77
4.3-a. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 1.2 exceed air	79
4.3-b. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 1.4 exceed air	81
4.3-c. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 1.6 exceed air	82
4.3-d. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 1.8 exceed air	84
4.3-e. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 2 exceed air	85
4.3-f. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 2.2 exceed air	86
4.3-g. The development of NO mass fraction of pollutant under different ratio of gaseous addition and 2.4 exceed air	88
4.4-a. The development of CO mass fraction under different ratio of gaseous addition and 1.2 exceed air	90
4.4-b. The development of CO mass fraction under different ratio of gaseous addition and 1.4 exceed air	91
4.4-c. The development of CO mass fraction under different ratio of gaseous addition and 1.6 exceed air	92
4.4-d. The development of CO mass fraction under different ratio of gaseous addition and 1.8 exceed air	93
4.4-e. The development of CO mass fraction under different ratio of gaseous addition and 2 exceed air	94
4.4-f. The development of CO mass fraction under different ratio of gaseous addition and 2.2 exceed air	95
4.4-g. The development of CO mass fraction under different ratio of gaseous addition and 2.4 exceed air	96

4.5-a. The development of CO ₂ mass fraction under different ratio of gaseous addition and 1.2 exceed air	99
4.5-b. The development of CO ₂ mass fraction under different ratio of gaseous addition and 1.4 exceed air	100
4.5-c. The development of CO ₂ mass fraction under different ratio of gaseous addition and 1.6 exceed air	101
4.5-d. The development of CO ₂ mass fraction under different ratio of gaseous addition and 1.8 exceed air	102
4.5-e. The development of CO ₂ mass fraction under different ratio of gaseous addition and 2 exceed air	103
4.5-f. The development of CO ₂ mass fraction under different ratio of gaseous addition and 2.2 exceed air	104
4.5-g. The development of CO ₂ mass fraction under different ratio of gaseous addition and 2.4 exceed air	105
A.1. Engine Specification	119
A.2. Gaseous Fuel Properties (20 C, 1 Bar)	119
A.3. Mass Fraction of H ₂ , CH ₄ , O ₂ and N ₂	120

LIST OF FIGURES

Figure		Page
2.1	The brake power versus A/F ratio for different H ₂ concentration and 1000 rpm (Ghazal, 2013b).	10
2.2	The brake power versus A/F ratio for different H ₂ concentration and 4000 rpm (Ghazal, 2013b).	10
2.3	Efficiency comparisons (Masood et al., 2007).	11
2.4	The mean effective pressure versus A/F ratio for different H ₂ concentration and 1000 rpm (Ghazal, 2013b).	12
2.5	The brake thermal efficiency versus A/F ratio for different H ₂ concentration and 4000 rpm (Ghazal, 2013b).	12
2.6	Effect of H ₂ addition on combustion duration, 70% load (Liew et al., 2010).	13
2.7	Effect of the addition of H ₂ and engine load on combustion duration (Liew et al., 2010).	14
2.8	Effect of H ₂ addition on cylinder pressure, 70% load (Liew et al., 2010).	15
2.9	Effect of H ₂ addition on peak cylinder pressure and its phasing, 70% load (Liew et al., 2010).	16
2.10	Variation of cylinder pressure with crank angle at 30% hydrogen enrichment mixture at full load condition (Saravanan & Nagarajan, 2008).	16
2.11	Effect of H ₂ addition on heat release process, 70% Load (Liew et al., 2010).	18
2.12	Variation of heat release with crank angle at 30% hydrogen enrichment mixture at full load condition (Saravanan & Nagarajan, 2008).	18
2.13	Effect of H ₂ addition on peak heat release rate and its phasing, 70% load (Liew et al., 2010).	19
2.14	Effect of H ₂ addition and engine load on peak heat release rate (Liew et al., 2010).	20
2.15	The mean effective pressure versus A/F ratio for different H ₂ concentration and 1000 rpm (Ghazal, 2013b).	21
2.16	The mean effective pressure versus A/F ratio for different H ₂ concentration and 4000 rpm (Ghazal, 2013b).	21
2.17	Variation of brake mean effective pressure with relative air–fuel ratio (Choi et al., 2005).	22
2.18	Variation of SEC with load (Saravanan & Nagarajan, 2008).	23
2.19	Variation of brake specific fuel consumption with relative air–fuel ratio (Choi et al., 2005).	23
2.20	Variation of tailpipe HC vs load (Saravanan & Nagarajan, 2008).	24
2.21	Variation of total hydrocarbon emissions with relative air–fuel ratio (Choi et al., 2005).	25
2.22	Effect of injection angle on PM (Masood et al., 2007).	27
2.23	Variation of smoke with load (Saravanan & Nagarajan, 2008).	28
2.24	Variation of particulate emissions with load (Saravanan & Nagarajan, 2008).	28
2.25	Variation of NO _x emissions with relative air–fuel ratio (Choi et al., 2005).	30
2.26	Variation of NO _x with load (Saravanan & Nagarajan, 2008).	31
2.27	NO _x reduction versus hydrogen supply at the exhaust pipe for the different levels of EGR ratio (Shin et al., 2011).	33
3.1	Steps of CFD analysis.	46

3.2	Computation Tools (HP)	48
3.3	Computational approach flow chart	49
3.4	Geometry and Mesh at TDC	50
3.5	Interface of computational grid	51
3.6	Computational geometry with defined zones	51
3.7	Motion of intake valve (red), exhaust valve (yellow), piston-full (green), and piston-limit (black) as a function of the crank angle.	52
3.8	Re-meshing method applied for computational model	53
3.9	First geometry and modified geometry with layering method	54
3.10	Geometry at TDC and BDC	54
3.11	Iteration convergence achieved from the simulation	57
4.1	(a) medium (11761 cells) and (b) fine sector meshes (19058 cells) (at TDC) used in 2D-CFD simulations.	61
4.2	Calculated in-cylinder pressure results using mesh a and b for engine speed 2000 rpm.	61
4.3	Validation of 2D simulation at 2000 rpm engine operation mode for dual fuel diesel for In-cylinder pressure.	62
4.4	Effect of different ratio of gaseous addition on peak in-cylinder pressure.	64
4.5-a	In-cylinder pressure curves at different ratio of gaseous addition and 1.2 exceed air.	64
4.5-b	In-cylinder pressure curves at different ratio of gaseous addition and 1.4 exceed air.	65
4.5-c	In-cylinder pressure curves at different ratio of gaseous addition and 1.6 exceed air.	66
4.5-d	In-cylinder pressure curves at different ratio of gaseous addition and 1.8 exceed air.	67
4.5-e	In-cylinder pressure curves at different ratio of gaseous addition and 2 exceed air.	68
4.5-f	In-cylinder pressure curves at different ratio of gaseous addition and 2.2 exceed air.	68
4.5-g	In-cylinder pressure curves at different ratio of gaseous addition and 2.4 exceed air.	69
4.6	Effect of different ratio of gaseous addition on peak in-cylinder temperature.	70
4.7-a	Temperature curves under different ratio of gaseous addition and 1.2 exceed air.	71
4.7-b	Temperature curves under different ratio of gaseous addition and 1.4 exceed air.	72
4.7-c	Temperature curves under different ratio of gaseous addition and 1.6 exceed air.	73
4.7-d	Temperature curves under different ratio of gaseous addition and 1.8 exceed air.	74
4.7-e	Temperature curves under different ratio of gaseous addition and 2 exceed air.	75
4.7-f	Temperature curves under different ratio of gaseous addition and 2.2 exceed air.	76
4.7-g	Temperature curves under different ratio of gaseous addition and 2.4 exceed air.	77
4.8	Effect of different ratio of gaseous addition on peak NO emissions.	78
4.9-a	NO emissions curves under different ratio of gaseous addition and 1.2 exceed air.	79
4.9-b	NO emissions curves under different ratio of gaseous addition and 1.4 exceed air.	80
4.9-c	NO emissions curves under different ratio of gaseous addition and 1.6 exceed air.	82

	exceed air.	
4.9-d	NO emissions curves under different ratio of gaseous addition and 1.8 exceed air.	83
4.9-e	NO emissions curves under different ratio of gaseous addition and 2 exceed air.	85
4.9-f	NO emissions curves under different ratio of gaseous addition and 2.2 exceed air.	86
4.9-g	NO emissions curves under different ratio of gaseous addition and 2.4 exceed air.	87
4.10	Effect of different ratio of gaseous addition on peak CO emissions	89
4.11-a	CO emissions curves under different ratio of gaseous addition and 1.2 exceed air.	90
4.11-b	CO emissions curves under different ratio of gaseous addition and 1.4 exceed air.	91
4.11-c	CO emissions curves under different ratio of gaseous addition and 1.6 exceed air.	92
4.11-d	CO emissions curves under different ratio of gaseous addition and 1.8 exceed air.	93
4.11-e	CO emissions curves under different ratio of gaseous addition and 2 exceed air.	94
4.11-f	CO emissions curves under different ratio of gaseous addition and 2.2 exceed air.	95
4.11-g	CO emissions curves under different ratio of gaseous addition and 2.4 exceed air.	96
4.12	Effect of different ratio of gaseous addition on peak CO ₂ emissions	98
4.13-a	CO ₂ emissions curves under different ratio of gaseous addition and 1.2 exceed air.	98
4.13-b	CO ₂ emissions curves under different ratio of gaseous addition and 1.4 exceed air.	99
4.13-c	CO ₂ emissions curves under different ratio of gaseous addition and 1.6 exceed air.	100
4.13-d	CO ₂ emissions curves under different ratio of gaseous addition and 1.8 exceed air.	101
4.13-e	CO ₂ emissions curves under different ratio of gaseous addition and 2 exceed air.	102
4.13-f	CO ₂ emissions curves under different ratio of gaseous addition and 2.2 exceed air.	103
4.13-g	CO ₂ emissions curves under different ratio of gaseous addition and 2.4 exceed air.	104

LIST OF ABBREVIATIONS

t	t F
	2 3 4 5
1	6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
8	101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116 117 118 119 120 121 122 123 124 125 126 127 128 129 130 131 132 133 134 135 136 137 138 139 140 141 142 143 144 145 146 147 148 149 150 151 152 153 154 155 156 157 158 159 160 161 162 163 164 165 166 167 168 169 170 171 172 173 174 175 176 177 178 179 180 181 182 183 184 185 186 187 188 189 190 191 192 193 194 195 196 197 198 199 200
6	201 202 203 204 205 206 207 208 209 210 211 212 213 214 215 216 217 218 219 220 221 222 223 224 225 226 227 228 229 230 231 232 233 234 235 236 237 238 239 240 241 242 243 244 245 246 247 248 249 250 251 252 253 254 255 256 257 258 259 260 261 262 263 264 265 266 267 268 269 270 271 272 273 274 275 276 277 278 279 280 281 282 283 284 285 286 287 288 289 290 291 292 293 294 295 296 297 298 299 300
6	301 302 303 304 305 306 307 308 309 310 311 312 313 314 315 316 317 318 319 320 321 322 323 324 325 326 327 328 329 330 331 332 333 334 335 336 337 338 339 340 341 342 343 344 345 346 347 348 349 350 351 352 353 354 355 356 357 358 359 360 361 362 363 364 365 366 367 368 369 370 371 372 373 374 375 376 377 378 379 380 381 382 383 384 385 386 387 388 389 390 391 392 393 394 395 396 397 398 399 400

CHAPTER 1

INTRODUCTION

1.1 Background

Diesel engines, , are commonly used both on and off roads due to their low Hydro Carbon (HC), high thermal efficiency and Carbon Monoxide (CO) emissions. On the other hand, they are a major contributor in terms of Nitric Oxides (NO_x) emissions as well as in terms of their Particulate Matter (PM). There have been many methods applied to reduce emissions. Diesel Particulate Filter (DPF) and Selective Catalytic Reduction (SCR) were used to reduce PM and NO_x emissions, respectively. As devices and catalysts are tough in retrofitting the engines of vehicles, these methods are based on the use of precious and expensive metals. As such, many compromising methods were put forward which also include the Dual-Fuel-Combustion (Sahoo et al., 2009).

Spark-ignition engines use mostly natural gas (NG). In diesel engines, the NG is applied under dieseling dual-fuel operation (Korakianitis et al., 2011; Papagiannakis & Hountalas, 2004). Papagiannakis et al. (Papagiannakis & Hountalas, 2004; Papagiannakis, Rakopoulos et al., 2010) worked on a diesel-NG dual-fuel single-cylinder diesel engine. The study outcome showed that there was an extension in the ignition delay of diesel-NG dual-fuel operation as compared with the usual diesel fuel operation. The highest rate of heat release and in-cylinder pressure was lowered as the NG addition was increased at low to middle loads, but it was raised at high load because of the enhanced rate of burning of the diesel-NG cooperated combustion. The CO/HC rapid increase and reduction of particulate paved way for the trade-off impact for the diesel-NG dual-fuel engine. CO emission control could be fulfilled via the intake air pre-heating and increasing the amount of the pilot diesel (Papagiannakis, Kotsiopoulos et al., 2010). NO emission's slight reduction was also seen. Poompipatpong and Cheenkachorn (Poompipatpong & Cheenkachorn, 2011) emphasized on the impact on the emissions of a 4-cylinder diesel-CNG dual-fuel engine by the compression ratio and with the engine speed. It was found in their experiment that the increased compression ratio and increased engine speed could attain the increased thermal efficiency and reduced emission of CO. Inferior thermal efficiency was seen when the engine load was low. In diesel fuel and natural gas together with the fuelled diesel engines, the major limitations are Undesirable thermal efficiency and much higher CO/HC emissions at low to middle loads (Korakianitis et al., 2011).

For the internal combustion engines, another best alternative fuel is hydrogen. This is due to its ability to enhance the engine efficiency as well as the emission reduction. Diesel-H₂ dual-fuel engine has gained considerable momentum in recent years (Bose & Banerjee, 2012; Liew et al., 2010). The process of diesel-H₂ dual-fuel combustion in one heavy-duty diesel engine was studied by Liew et al. (Liew et al., 2010). It was shown that the peak in cylinder pressure drastically

increased at 70 percent of full engine load and this influence was to be under control for the purpose of safety as well as for the engine's mechanical durability. It was observed that hydrogen's combustion efficiency was considerably less when a small quantity of hydrogen was added. Gatts et al. (Gatts et al., 2010) also explored on the combustion efficiency of hydrogen through the measurement of unburned hydrogen in the exhaust gas. It was shown by these studies that the engine load was dependent on the hydrogen combustion efficiency and the hydrogen must be added at high load for attaining high energy conversion efficiency for the hydrogen fuel as well as for the diesel fuel. Results of Liew et al. and Lilik et al. (Liew et al., 2012; Lilik et al., 2010) indicated that HC/CO/CO₂/PM emissions reduced almost linearly when the hydrogen addition was increased, which indicates that the reductions in carbon-based gaseous as well as particle emissions are associated with hydrogen quantity being added. NO_x emission on the other hand, decreased at low to middle loads, whereas it increased at high load because of the rapid burning rate of hydrogen which resulted in high combustion temperature as well as improved the formation of NO_x. The thermal efficiency is based on the load and speed of engine, as well as on hydrogen quantity being added which was shown in the study conducted by Miyamoto et al. (Miyamoto et al., 2011). Kumar Bose and Banerjee (Bose & Banerjee, 2012) explored the hydrogen addition's impact on emission reduction as well as performance trade-off with hot and cooled EGRs. They showed that 10 percent and 20 percent of EGR indicated a powerful potential in the reduction of NO_x as well as smoke emissions. This also maintained a simultaneous reduction on HC/CO/CO₂/BSFC together with the gain of sustainability on the Brake Thermal Efficiency (BTE).

In the case of conventional gas-diesel dual-fuel engines, there is an engine efficiency sacrifice and higher level of some of the emissions such as that of HC/CO which cannot be solved without using the after-treatment equipment. Some new methods have to be developed. Hydrogen enrichment has the ability to enhance the process of combustion of some gases fuels including LPG and NG, so as to enhance the total gases fuels energy usage efficiency. Prior times, using the mixture of CNG- Hydrogen for spark-ignition fuel engine has been explored much in studies (Acikgoz & Celik, 2012; Mariani et al., 2013) and is shown to be good for the elimination of some negative aspects of NG spark-ignition engines (Korakianitis et al., 2011). The enhancement is related with the unique characteristics of hydrogen, including its broad flammability, rapid burning velocity, less ignition energy and its carbon-free nature. In the recent times, for increasing the performance of conventional gas-diesel dual-fuel engines, studies started focusing on tri-fuel engines. Lata et al. (Lata & Misra, 2011; Lata et al., 2012) did a series of in-depth experimental and theoretical studies on diesel engines which use hydrogen-LPG mixture as the gaseous fuel, along with the diesel fuel. The main results of these studies showed that low efficiency at low load for LPG-diesel dual-fuel mode was enhanced with the hydrogen enrichment while the engine was functioning at above 10 percent of the full load. Similar to this, methane has a low flame propagation speed as well as slight flammability whereas hydrogen has the extreme opposite characteristics. As such, adding hydrogen can enhance methane's combustion process making it extra convenient

in diesel engine applications. For hydrogen-diesel dual-fuel mode, the rapid burning rate, increased diffusivity and reduced hydrogen ignition energy enable the combustion to become unstable, particularly at increased engine loads that might result in knocking. The knocking is harmful to engine's mechanical durability as well as safety, as discussed earlier. Methane enrichment has the ability to make the combustion of hydrogen stable and smoother which can prevent imperfect combustion. Methane can also lower the combustion temperature of hydrogen so as to repress NO_x emission. This study focuses on comparing the combustion characteristics as well as emission features of a diesel engine under diesel- CH_4 and diesel- H_2 dual-fuel operations, as well as under diesel- CH_4 - H_2 tri-fuel operations, where the gaseous fuels substitute up to 50 percent of the overall fuel. Under the tri-fuel operation, hydrogen and methane are blended with different percent fractions and then the 2 gaseous fuels are combusted together with the diesel fuel.

1.2 Problem Statement

For conventional gas-diesel dual-fuel engines, there is an engine efficiency sacrifice and higher level of some of the emissions such as that of HC/CO which cannot be solved without using the after-treatment equipment. This equipment however is expensive. Therefore, novel method has to be developed. Adding hydrogen can enhance the process of combustion of some gaseous fuels including LPG and NG, so as to enhance the total gases fuels energy usage efficiency. Flame propagation speed of methane is low and also it has narrow flammability. Meanwhile, the hydrogen has contrary traits; therefore hydrogen enrichment can enhance the process of methane combustion and make its convenient for application of diesel engine. For hydrogen-diesel dual-fuel mode, rapid burning rate, increased diffusivity as well as low ignition energy hydrogen increased the combustion abnormal characteristics such as knocking. The knocking is harmful to the mechanical durability as well as safety of engine. On the other hand, a hydrogen affects emissions by reducing the hydrocarbon (HC), CO, CO_2 , PM, and smoke. The effective method to solve these problems is to blend H_2 , CH_4 , and diesel to meet the characteristics required by the engine. Methane addition makes hydrogen combustion smoother as well as much more stable and blocks abnormal combustion. Methane can also lower the combustion temperature of hydrogen so as to repress NO_x emission.

1.3 Hypothesis

The current study proposes that by adding hydrogen into CH_4 fuel and diesel it can improve velocity of combustion and therefore enhancing the combustion characteristics. Mix of natural gas with hydrogen is required to increase the lean-burn attributes as well as minimize the actual engines emissions (mainly CO_2 , HC as well as CO), but the possibility involving greater NO_x emissions will be involving concern. This helps combustion behaviour of action with the possibility to formulate engines with increased performance and lower environmentally friendly impact. Hydrogen itself provides the possible to be an alternate to be able to regular fuel, since it is absolutely carbon-free along with not too difficult to

make but high pricey. The employment of NG/hydrogen mixtures comprising H₂ gives good possibility to offer the rewards associated with the particular hydrogen without having large modification involving currently existing CNG engine.

1.4 Research Objectives

The aim of the current study is to simulate dual and tri fuel diesel engines consist of methane, hydrogen, and diesel. The current study also observes the impact of mixing ratios with variation of exceed air. The specific objectives are as follow:

1. To perform CFD simulation on CH₄-diesel and H₂-diesel for dual-fuel mode and on CH₄-H₂-diesel for tri-fuel modes.
2. To determine amount of gaseous addition for best condition.
3. To evaluate the combustion characteristics and emissions of a compression ignition engine with varying engine operations under different ratios of exceed air (λ).

1.5 Scope of Research

This study concentrates on the impact of combining tri fuels namely methane, hydrogen and diesel on combustion characteristic. Furthermore the effect of λ (exceed air) was looked into at each of the engine operations both dual and tri-modes. This study has the scope to deliver combustion characteristics and emissions. There has also been an attempt to illustrate the engine's combustion chamber using 2 dimensional analyses, hence enabling better comprehension of the behaviour of combustion chamber.

1.6 Thesis Layout

This thesis has been divided orderly into five chapters, the thesis starts with introduction in Chapter 1 which includes a background of dual and tri-fuel diesel engine.

Chapter 2 explains benefits for using alternative fuels of diesel engines and explains the effect of hydrogen addition on performance and combustion as well as emissions in direct diesel engines.

Chapter 3 elaborates on the methodology used that includes a description of the grid generation of diesel engine using the Gambit software that created the needed mesh by moving the dynamic mesh or MDM model and also defines the conditions of the boundary and sets the solver variables in the Fluent software.

Chapter 4 illuminates the results which were accomplished from CFD simulation as well as the corresponding discussions.

Chapter 5 displays the recommendation for future studies and final conclusion of this project.

REFERENCES

- Abagnale, C., Cameretti, M., De Simio, L., Gambino, M., Iannaccone, S., & Tuccillo, R. (2014). Numerical simulation and experimental test of dual fuel operated diesel engines. *Applied Thermal Engineering*, 65(1), 403-417.
- Abdullah, S., Kurniawan, W. H., & Shamsudeen, A. (2008). Numerical analysis of the combustion process in a compressed natural gas direct injection engine. *Journal of Applied Fluid Mechanics*, 1(2), 65-86.
- Aceves, S. M., Flowers, D. L., Martinez-Frias, J., Smith, J. R., Dibble, R., Au, M., & Girard, J. (2001). *HCCI Combustion: Analysis and Experiments*,
- Acikgoz, B., & Celik, C. (2012). An experimental study on performance and emission characteristics of a methane–hydrogen fuelled gasoline engine. *International Journal of Hydrogen Energy*, 37(23), 18492-18497.
- Adnan, R., Masjuki, H., & Mahlia, T. (2009). An experimental investigation of unmodified DI diesel engine with hydrogen addition. *Energy and Environment, 2009. ICEE 2009. 3rd International Conference On*, 45-49.
- Adnan, R., Masjuki, H., & Mahlia, T. (2012). Performance and emission analysis of hydrogen fueled compression ignition engine with variable water injection timing. *Energy*, 43(1), 416-426.
- Akansu, S. O., Dulger, Z., Kahraman, N., & Veziroğlu, T. N. (2004). Internal combustion engines fueled by natural gas—hydrogen mixtures. *International Journal of Hydrogen Energy*, 29(14), 1527-1539.
- Akansu, S. O., Kahraman, N., & Ceper, B. (2007). Experimental study on a spark ignition engine fuelled by methane–hydrogen mixtures. *International Journal of Hydrogen Energy*, 32(17), 4279-4284.
- Alkidas, A. C. (2007). Combustion advancements in gasoline engines. *Energy Conversion and Management*, 48(11), 2751-2761.
- Alrazen, H. A., Talib, A. A., Adnan, R., & Ahmad, K. (2016). A review of the effect of hydrogen addition on the performance and emissions of the compression–Ignition engine. *Renewable and Sustainable Energy Reviews*, 54, 785-796.
- Antunes, J. G., Mikalsen, R., & Roskilly, A. (2009). An experimental study of a direct injection compression ignition hydrogen engine. *International Journal of Hydrogen Energy*, 34(15), 6516-6522.
- Aung, K., Hassan, M., & Faeth, G. (1998). Effects of pressure and nitrogen dilution on flame/stretch interactions of laminar premixed H₂/O₂/N₂ flames. *Combustion and Flame*, 112(1), 1-15.

- Bauer, C., & Forest, T. (2001). Effect of hydrogen addition on the performance of methane-fueled vehicles. part I: Effect on SI engine performance. *International Journal of Hydrogen Energy*, 26(1), 55-70.
- Bedford, F., Hu, X., & Schmidt, U. (2004). In-cylinder combustion modeling and validation using fluent. *14th Annual International Multidimensional Engine Modeling User's Group Meeting, Detroit*,
- Boretti, A. (2011). Advantages of the direct injection of both diesel and hydrogen in dual fuel H₂ ICE. *International Journal of Hydrogen Energy*, 36(15), 9312-9317.
- Bose, P. K., & Banerjee, R. (2012). An experimental investigation on the role of hydrogen in the emission reduction and performance trade-off studies in an existing diesel engine operating in dual fuel mode under exhaust gas recirculation. *Journal of Energy Resources Technology*, 134(1), 012601.
- Bradley, D., Lawes, M., Liu, K., Verhelst, S., & Woolley, R. (2007). Laminar burning velocities of lean hydrogen–air mixtures at pressures up to 1.0 MPa. *Combustion and Flame*, 149(1), 162-172.
- Breshears, R., Cotrill, H., & Rupe, J. (1973). Partial hydrogen injection into internal combustion engines effect on emissions and fuel economy. *First Symposium on Low Pollution Power Systems Development, Ann Arbor, Michigan*,
- Bression, G., Soleri, D., Savy, S., Dehoux, S., Azoulay, D., Hamouda, H. B., . . . Lawrence, N. (2008). *A Study of Methods to Lower HC and CO Emissions in Diesel HCCI*,
- Cadman, W., & Johnson, J. H. (1986). *The Study of the Effect of Exhaust Gas Recirculation on Engine Wear in a Heavy-Duty Diesel Engine using Analytical Ferrography*,
- Carlucci, A., De Risi, A., Laforgia, D., & Naccarato, F. (2008). Experimental investigation and combustion analysis of a direct injection dual-fuel diesel–natural gas engine. *Energy*, 33(2), 256-263.
- Chintala, V., & Subramanian, K. (2015). An effort to enhance hydrogen energy share in a compression ignition engine under dual-fuel mode using low temperature combustion strategies. *Applied Energy*, 146, 174-183.
- Cho, H. M., & He, B. (2007). Spark ignition natural gas engines—A review. *Energy Conversion and Management*, 48(2), 608-618.
- Choi, G. H., Chung, Y. J., & Han, S. B. (2005). Performance and emissions characteristics of a hydrogen enriched LPG internal combustion engine at 1400rpm. *International Journal of Hydrogen Energy*, 30(1), 77-82.

- Colin R., Ferguson, & Kirkpatrick, A. T. (2001). *Internal combustion engines: Applied thermosciences* John Wiley & Sons.
- Das, L. (1996). Utilization of hydrogen-cng blend in internal combustion engine. *HYDROGEN ENERGY PROGRESS*, 2, 1513-1536.
- Das, L. (2002). Hydrogen engine: Research and development (R&D) programmes in indian institute of technology (IIT), delhi. *International Journal of Hydrogen Energy*, 27(9), 953-965.
- de Moraes, A. M., Justino, M. A. M., Valente, O. S., de Moraes Hanriot, S., & Sodr , J. R. (2013). Hydrogen impacts on performance and CO 2 emissions from a diesel power generator. *International Journal of Hydrogen Energy*, 38(16), 6857-6864.
- Dimopoulos, P., Bach, C., Soltic, P., & Boulouchos, K. (2008). Hydrogen–natural gas blends fuelling passenger car engines: Combustion, emissions and well-to-wheels assessment. *International Journal of Hydrogen Energy*, 33(23), 7224-7236.
- Fayaz, H., Saidur, R., Razali, N., Anuar, F., Saleman, A., & Islam, M. (2012). An overview of hydrogen as a vehicle fuel. *Renewable and Sustainable Energy Reviews*, 16(8), 5511-5528.
- Fern ndez-Galisteo, D., S nchez, A., Li n n, A., & Williams, F. (2009). One-step reduced kinetics for lean hydrogen–air deflagration. *Combustion and Flame*, 156(5), 985-996.
- Fluent, F. (2006). 6.3 user’s guide. *Fluent Inc*,
- Ganesh, D., Nagarajan, G., & Ibrahim, M. M. (2008). Study of performance, combustion and emission characteristics of diesel homogeneous charge compression ignition (HCCI) combustion with external mixture formation. *Fuel*, 87(17), 3497-3503.
- Gatts, T., Li, H., Liew, C., Liu, S., Spencer, T., Wayne, S., & Clark, N. (2010). An experimental investigation of H 2 emissions of a 2004 heavy-duty diesel engine supplemented with H 2. *International Journal of Hydrogen Energy*, 35(20), 11349-11356.
- Gatts, T., Liu, S., Liew, C., Ralston, B., Bell, C., & Li, H. (2012). An experimental investigation of incomplete combustion of gaseous fuels of a heavy-duty diesel engine supplemented with hydrogen and natural gas. *International Journal of Hydrogen Energy*, 37(9), 7848-7859.
- Ghazal, O. H. (2013a). A comparative evaluation of the performance of different fuel induction techniques for blends hydrogen–methane SI engine. *International Journal of Hydrogen Energy*, 38(16), 6848-6856.

- Ghazal, O. H. (2013b). Performance and combustion characteristic of CI engine fueled with hydrogen enriched diesel. *International Journal of Hydrogen Energy*, 38(35), 15469-15476.
- Ghili. (2011). *Computational fluid dynamic analysis of knock onset in diesel dual fuel engine* (Master Thesis).
- Hairuddin, A. A., Yusaf, T., & Wandel, A. P. (2014). A review of hydrogen and natural gas addition in diesel HCCI engines. *Renewable and Sustainable Energy Reviews*, 32, 739-761.
- Hallgren, B. E. (2000). *Effects of Oxygenated Fuels on DI Diesel Combustion and Emissions*,
- Heywood, J. B. (1988). *Internal combustion engine fundamentals* McGraw-hill New York.
- Hoekstra, R. L., Van Blarigan, P., & Mulligan, N. (1996). *NO_x Emissions and Efficiency of Hydrogen, Natural Gas, and Hydrogen/Natural Gas Blended Fuels*,
- Houseman, J., & Hoehn, F. W. (1974). *A Two-Charge Engine Concept: Hydrogen Enrichment*,
- Jayashankara, B., & Ganesan, V. (2010). Effect of fuel injection timing and intake pressure on the performance of a DI diesel engine—A parametric study using CFD. *Energy Conversion and Management*, 51(10), 1835-1848.
- Jones, W. P., & Tyliszczak, A. (2010). Large eddy simulation of spark ignition in a gas turbine combustor. *Flow, Turbulence and Combustion*, 85(3-4), 711-734.
- Jones, W., & Lindstedt, R. (1988). Global reaction schemes for hydrocarbon combustion. *Combustion and Flame*, 73(3), 233-249.
- Karavalakis, G., Hajbabaie, M., Durbin, T. D., Johnson, K. C., Zheng, Z., & Miller, W. J. (2013). The effect of natural gas composition on the regulated emissions, gaseous toxic pollutants, and ultrafine particle number emissions from a refuse hauler vehicle. *Energy*, 50, 280-291.
- Karim, G. A. (2003). Hydrogen as a spark ignition engine fuel. *International Journal of Hydrogen Energy*, 28(5), 569-577.
- Kayes, D., & Hochgreb, S. (1999). Mechanisms of particulate matter formation in spark-ignition engines. 1. effect of engine operating conditions. *Environmental Science & Technology*, 33(22), 3957-3967.
- Khan, I., Greeves, G., & Wang, C. (1973). *Factors Affecting Smoke and Gaseous Emissions from Direct Injection Engines and a Method of Calculation*,

- Khan, M. I., Yasmin, T., & Shakoor, A. (2015). Technical overview of compressed natural gas (CNG) as a transportation fuel. *Renewable and Sustainable Energy Reviews*, 51, 785-797.
- Knop, V., Benkenida, A., Jay, S., & Colin, O. (2008). Modelling of combustion and nitrogen oxide formation in hydrogen-fuelled internal combustion engines within a 3D CFD code. *International Journal of Hydrogen Energy*, 33(19), 5083-5097.
- Komninos, N. (2009). Investigating the importance of mass transfer on the formation of HCCI engine emissions using a multi-zone model. *Applied Energy*, 86(7), 1335-1343.
- Kong, S., & Reitz, R. D. (2003). Numerical study of premixed HCCI engine combustion and its sensitivity to computational mesh and model uncertainties. *Combustion Theory and Modelling*, 7(2), 417-433.
- Konnov, A. A. (2008). Remaining uncertainties in the kinetic mechanism of hydrogen combustion. *Combustion and Flame*, 152(4), 507-528.
- Korakianitis, T., Namasivayam, A., & Crookes, R. (2011). Natural-gas fueled spark-ignition (SI) and compression-ignition (CI) engine performance and emissions. *Progress in Energy and Combustion Science*, 37(1), 89-112.
- Kuo, K. K. (1986). Principles of combustion.
- Lata, D., & Misra, A. (2011). Analysis of ignition delay period of a dual fuel diesel engine with hydrogen and LPG as secondary fuels. *International Journal of Hydrogen Energy*, 36(5), 3746-3756.
- Lata, D., Misra, A., & Medhekar, S. (2012). Effect of hydrogen and LPG addition on the efficiency and emissions of a dual fuel diesel engine. *International Journal of Hydrogen Energy*, 37(7), 6084-6096.
- Law, C., & Kwon, O. (2004). Effects of hydrocarbon substitution on atmospheric hydrogen-air flame propagation. *International Journal of Hydrogen Energy*, 29(8), 867-879.
- Lee, S., Yi, H., & Kim, E. (1995). Combustion characteristics of intake port injection type hydrogen fueled engine. *International Journal of Hydrogen Energy*, 20(4), 317-322.
- Liew, C., Li, H., Liu, S., Besch, M., Ralston, B., Clark, N., & Huang, Y. (2012). Exhaust emissions of a H₂-enriched heavy-duty diesel engine equipped with cooled EGR and variable geometry turbocharger. *Fuel*, 91(1), 155-163.
- Liew, C., Li, H., Nuszowski, J., Liu, S., Gatts, T., Atkinson, R., & Clark, N. (2010). An experimental investigation of the combustion process of a heavy-

- duty diesel engine enriched with H₂. *International Journal of Hydrogen Energy*, 35(20), 11357-11365.
- Lilik, G. K., Zhang, H., Herreros, J. M., Haworth, D. C., & Boehman, A. L. (2010). Hydrogen assisted diesel combustion. *International Journal of Hydrogen Energy*, 35(9), 4382-4398.
- Liu, A. B., Mather, D., & Reitz, R. D. (1993). *Modeling the Effects of Drop Drag and Breakup on Fuel Sprays*,
- López, J. M., Gómez, Á., Aparicio, F., & Sanchez, F. J. (2009). Comparison of GHG emissions from diesel, biodiesel and natural gas refuse trucks of the city of madrid. *Applied Energy*, 86(5), 610-615.
- Márcio de Almeida, D., & Ribeiro, S. K. (2009). Assessing total and renewable energy in brazilian automotive fuels. A life cycle inventory (LCI) approach. *Renewable and Sustainable Energy Reviews*, 13(6), 1326-1337.
- Mariani, A., Prati, M. V., Unich, A., & Morrone, B. (2013). Combustion analysis of a spark ignition ic engine fuelled alternatively with natural gas and hydrogen-natural gas blends. *International Journal of Hydrogen Energy*, 38(3), 1616-1623.
- Masood, M., Ishrat, M., & Reddy, A. (2007). Computational combustion and emission analysis of hydrogen–diesel blends with experimental verification. *International Journal of Hydrogen Energy*, 32(13), 2539-2547.
- Miyamoto, T., Hasegawa, H., Mikami, M., Kojima, N., Kabashima, H., & Urata, Y. (2011). Effect of hydrogen addition to intake gas on combustion and exhaust emission characteristics of a diesel engine. *International Journal of Hydrogen Energy*, 36(20), 13138-13149.
- Naber, J. D., & Szwaja, S. (2007). Statistical approach to characterize combustion knock in the hydrogen fuelled SI engine. *Journal of KONES*, 14, 443-450.
- Najjar, Y. S. (2013). Hydrogen safety: The road toward green technology. *International Journal of Hydrogen Energy*, 38(25), 10716-10728.
- Ng, H. K., Gan, S., Ng, J., & Pang, K. M. (2013). Simulation of biodiesel combustion in a light-duty diesel engine using integrated compact biodiesel–diesel reaction mechanism. *Applied Energy*, 102, 1275-1287.
- Oilgae – Glossary. (2014). Retrieved from http://www.oilgae.com/ref/glos/nox_emissions.html
- Papagiannakis, R., & Hountalas, D. (2004). Combustion and exhaust emission characteristics of a dual fuel compression ignition engine operated with pilot diesel fuel and natural gas. *Energy Conversion and Management*, 45(18), 2971-2987.

- Papagiannakis, R., Kotsiopoulos, P., Zannis, T., Yfantis, E., Hountalas, D., & Rakopoulos, C. (2010). Theoretical study of the effects of engine parameters on performance and emissions of a pilot ignited natural gas diesel engine. *Energy*, 35(2), 1129-1138.
- Papagiannakis, R., Rakopoulos, C., Hountalas, D., & Rakopoulos, D. (2010). Emission characteristics of high speed, dual fuel, compression ignition engine operating in a wide range of natural gas/diesel fuel proportions. *Fuel*, 89(7), 1397-1406.
- Poompipatpong, C., & Cheenkachorn, K. (2011). A modified diesel engine for natural gas operation: Performance and emission tests. *Energy*, 36(12), 6862-6866.
- Ramamurthy, B. (2006). Design of a catalyst system with periodic flow reversal for lean burn natural gas engines.
- Raviteja, S., & Kumar, G. (2015). Effect of hydrogen addition on the performance and emission parameters of an SI engine fueled with butanol blends at stoichiometric conditions. *International Journal of Hydrogen Energy*, 40(30), 9563-9569.
- Rogg, B. (1993). *Reduced kinetic mechanisms for applications in combustion systems* Springer Science & Business Media.
- Rottengruber, H., Berckmueller, M., Elsaesser, G., Brehm, N., & Schwarz, C. (2004). Operation strategies for hydrogen engines with high power density and high efficiency. *15th Annual US Hydrogen Conference*,
- Sahoo, B., Sahoo, N., & Saha, U. (2009). Effect of engine parameters and type of gaseous fuel on the performance of dual-fuel gas diesel engines—A critical review. *Renewable and Sustainable Energy Reviews*, 13(6), 1151-1184.
- Santoso, W., Bakar, R., & Nur, A. (2013). Combustion characteristics of diesel-hydrogen dual fuel engine at low load. *Energy Procedia*, 32, 3-10.
- Saravanan, N., & Nagarajan, G. (2008). An experimental investigation of hydrogen-enriched air induction in a diesel engine system. *International Journal of Hydrogen Energy*, 33(6), 1769-1775.
- Saravanan, N., & Nagarajan, G. (2010). An experimental investigation on hydrogen fuel injection in intake port and manifold with different EGR rates. *Energy Environ*, 1, 221-248.
- Saravanan, N., Nagarajan, G., Dhanasekaran, C., & Kalaiselvan, K. (2007). Experimental investigation of hydrogen port fuel injection in DI diesel engine. *International Journal of Hydrogen Energy*, 32(16), 4071-4080.

- Saravanan, N., Nagarajan, G., Kalaiselvan, K., & Dhanasekaran, C. (2008). An experimental investigation on hydrogen as a dual fuel for diesel engine system with exhaust gas recirculation technique. *Renewable Energy*, 33(3), 422-427.
- Saravanan, N., Nagarajan, G., & Narayanasamy, S. (2008). An experimental investigation on DI diesel engine with hydrogen fuel. *Renewable Energy*, 33(3), 415-421.
- Saxena, P., & Williams, F. A. (2006). Testing a small detailed chemical-kinetic mechanism for the combustion of hydrogen and carbon monoxide. *Combustion and Flame*, 145(1), 316-323.
- Scarcelli, R. (2008). *Lean-Burn Operation for Natural Gas/Air Mixtures: The Dual-Fuel Engines*,
- Schefer, R. W., Wicksall, D., & Agrawal, A. (2002). Combustion of hydrogen-enriched methane in a lean premixed swirl-stabilized burner. *Proceedings of the Combustion Institute*, 29(1), 843-851.
- Selle, L., Lartigue, G., Poinso, T., Koch, R., Schildmacher, K., Krebs, W., . . . Veynante, D. (2004). Compressible large eddy simulation of turbulent combustion in complex geometry on unstructured meshes. *Combustion and Flame*, 137(4), 489-505.
- Shin, B., Cho, Y., Han, D., Song, S., & Chun, K. M. (2011). Hydrogen effects on NO_x emissions and brake thermal efficiency in a diesel engine under low-temperature and heavy-EGR conditions. *International Journal of Hydrogen Energy*, 36(10), 6281-6291.
- Shojaeefard, M., & Noorpoor, A. (2008). Flow simulation in engine cylinder with spring mesh. *American Journal of Applied Sciences*, 5(10), 1336.
- Sjöberg, M., & Dec, J. E. (2005). An investigation into lowest acceptable combustion temperatures for hydrocarbon fuels in HCCI engines. *Proceedings of the Combustion Institute*, 30(2), 2719-2726.
- Spalding, D. (1971). Mixing and chemical reaction in steady confined turbulent flames. *Symposium (International) on Combustion*, , 13(1) 649-657.
- Stone, R. (1999). Introduction to internal combustion engines. *Basingstoke, UK*,
- Sun, Z., Liu, F., Liu, X., Sun, B., & Sun, D. (2012). Research and development of hydrogen fuelled engines in china. *International Journal of Hydrogen Energy*, 37(1), 664-681.
- Szwaja, S., & Grab-Rogalinski, K. (2009). Hydrogen combustion in a compression ignition diesel engine. *International Journal of Hydrogen Energy*, 34(10), 4413-4421.

- Tanaka, S., Ayala, F., & Keck, J. C. (2003). A reduced chemical kinetic model for HCCI combustion of primary reference fuels in a rapid compression machine. *Combustion and Flame*, 133(4), 467-481.
- Tira, H. S. (2013). *Impact of Alternative Fuels and Hydrogen-Enriched Gaseous Fuel on Combustion and Emissions in Diesel Engines*,
- Verhelst, S., & Sierens, R. (2007). A quasi-dimensional model for the power cycle of a hydrogen-fuelled ICE. *International Journal of Hydrogen Energy*, 32(15), 3545-3554.
- Verhelst, S., & Sierens, R. (2008). A two-zone thermodynamic model for hydrogen-fueled SI engines. *The 7th COMODIA International Conference on Modeling and Diagnostics for Advanced Engine Systems*, 773-778.
- Verhelst, S., & Wallner, T. (2009). Hydrogen-fueled internal combustion engines. *Progress in Energy and Combustion Science*, 35(6), 490-527.
- Wannatong, K., Akarapanyavit, N., Siengsanorh, S., & Chanchaona, S. (2007). *Combustion and Knock Characteristics of Natural Gas Diesel Dual Fuel Engine*,
- Welch, A., & Wallace, J. (1990). *Performance Characteristics of a Hydrogen-Fueled Diesel Engine with Ignition Assist*,
- White, C., Steeper, R., & Lutz, A. (2006). The hydrogen-fueled internal combustion engine: A technical review. *International Journal of Hydrogen Energy*, 31(10), 1292-1305.
- Williams, A. M. (2004). Lean NOx trap catalysis for lean burn natural gas engines.
- Wong, W. L. (2005). *Compressed Natural Gas as an Alternative Fuel in Diesel Engines*,
- Xu, J., Zhang, X., Liu, J., & Fan, L. (2010). Experimental study of a single-cylinder engine fueled with natural gas-hydrogen mixtures. *International Journal of Hydrogen Energy*, 35(7), 2909-2914.
- Yang, Z., Chu, C., Wang, L., & Huang, Y. (2015). Effects of H₂ addition on combustion and exhaust emissions in a diesel engine. *Fuel*, 139, 190-197.
- Yap, D., Peucheret, S., Megaritis, A., Wyszynski, M., & Xu, H. (2006). Natural gas HCCI engine operation with exhaust gas fuel reforming. *International Journal of Hydrogen Energy*, 31(5), 587-595.
- Yu, G., Law, C., & Wu, C. (1986). Laminar flame speeds of hydrocarbon air mixtures with hydrogen addition. *Combustion and Flame*, 63(3), 339-347.

- Zamel, N., & Li, X. (2006). Life cycle analysis of vehicles powered by a fuel cell and by internal combustion engine for canada. *Journal of Power Sources*, 155(2), 297-310.
- Zhao, H., Hu, J., & Ladommatos, N. (2000). In-cylinder studies of the effects of CO₂ in exhaust gas recirculation on diesel combustion and emissions. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of Automobile Engineering*, 214(4), 405-419.
- Zhou, J., Cheung, C., & Leung, C. (2014). Combustion, performance and emissions of a diesel engine with H₂, CH₄ and H₂-CH₄ addition. *International Journal of Hydrogen Energy*, 39(9), 4611-4621.