

# UNIVERSITI PUTRA MALAYSIA

DESIGN AND DEVELOPMENT OF KENAF HARVESTING MACHINE

**OMID GHAHRAEI** 

FK 2011 107

# DESIGN AND DEVELOPMENT OF KENAF HARVESTING MACHINE



By

**OMID GHAHRAEI** 



Thesis Submitted to the School of Graduate Studies, Universiti Putra Malaysia, in Fulfilment of the Requirements for the Degree of Doctor of Philosophy

August 2011

# Dedicated to

To my family members especially my beloved wife, my

dear son, and my ever-encouraging parents

for their love

Abstract of thesis presented to the Senate of Universiti Putra Malaysia in fulfilment of the requirement for the degree of Doctor of Philosophy

#### DESIGN AND DEVELOPMENT OF KENAF HARVESTING MACHINE

By

#### **OMID GHAHRAEI**

August 2011

Chair: Professor Desa Ahmad, PhD, P.Eng.

**Faculty: Engineering** 

Kenaf whole-stem having long bast fibres is more suitable for industries to produce textile and bio-composite panels. Current modified harvesters (sugarcane harvesters, forage harvesters, choppers, reaper binders, mower conditioners, pedestrian harvesters) normally chop the kenaf stems to small segments and/or crash the fibres or unable to cut the thick stems. So, a kenaf harvesting machine is required to cut the thick intact whole-stems having high capacity and satisfactory cutting quality with no damage on the bast fibres. Currently manual whole-stem harvesting which is a labour-intensive, time-consuming, and less profitable process, is being practiced in Malaysia.

In this study, a new pull-type four-row whole-stem kenaf harvesting machine with a rotary impact cutting system was designed, developed and evaluated. The machine cutting system design was based on effective cutting knife edge angle (ANE) and cutting rotational speed. In this research, specific cutting force (SCF) and specific

cutting energy (SCE) were measured by considering knife edge angle (ANE), shear angle (SA), knife approach angle (ANA), knife rake angle (ANR), and the cross-sectional area of plant stems. In addition, an experimental impact cutting machine was manufactured and tested in the field. The rotational speed obtained with this machine had the lowest cutting torque. Kenaf stems of the V36 variety were used as the experimental material. An analysis of variance of the SCF and SCE values of the kenaf stems showed that the effects of all the above-mentioned angles (considering a broad range) on SCF and SCE were significant. Moreover, the preferred values of ANE, SA, ANA, and ANR were 25°, 40°, 40°, and 40°, respectively, according to Duncan's multiple range test (DMRT). Based on the impact cutting test, the rotational cutting speed had a significant effect on the specific cutting torque. Increasing the rotational speed from 308 to 788 rpm decreased the cutting torque by 26.3%. The preferred rotational speed with a minimum cutting torque used in designing the cutting system was 712 rpm. The experimental impact cutting machine had an estimated effective field capacity of 0.56 ha/8 h day. The average moisture content of cut samples from the lower area of the stems was 70.78% (dry basis).

The harvesting machine operated best at the field speeds of 3-6 km/h resulted from the cutting quality tests and recommended by DMRT. In preliminary field tests, the average values of the effective filed capacity (EFC), field efficiency (FE), and material capacity (MC) of the machine were found to be 1.68 ha/8 h day, 70.6%, and 114.8 t/8 h day for single-row harvesting (with 75 cm row spacing and about 20 stems/m of row) and 3.37 ha/8 h day, 74%, and 241.9 t/8 h day for 2-row harvesting

(with 75 cm row spacing and about 20 stems/m of row), respectively at recommended speeds of 3-6 km/h. The average expected values of EFC, FE, and MC of the machine for 4-row harvesting (with 30 cm row spacing and 10 stems/m of row) were foreseen to be 2.92 ha/8 h day, 77%, and 249 t/8 h day, respectively at recommended speeds of 3-6 km/h in standard field conditions (planted by an accurate planter, proper watering and fertilization, and with no weed or grass). Maximum height, average diameter, average cutting height, and average moisture content of the kenaf stems at the harvesting time were measured as 3.10 m, 21.8 mm, 20 cm, and 71.8% (dry basis), respectively.

The highest recommended machine effective field capacity evaluated in this study was capable of replacing up to 370 persons per day when harvesting by traditional hand methods. Based on cost analysis results, the total manual harvesting operation cost was 32 times more than the total mechanical harvesting operation cost for harvesting 1,500 hectares of Malaysia kenaf fields for fibre production for one time plantation a year in 2010.

Abstrak tesis yang dikemukakan kepada Senat Universiti Putra Malaysia sebagai memenuhi keperluan untuk ijazah Doktor Falsafah

#### **REKABENTUK DAN PEMBINAAN MESIN PENUAI KENAF**

Oleh

#### **OMID GHAHRAEI**

**Ogos 2011** 

Pengerusi: Professor Desa Ahmad, PhD, P.Eng.

#### Fakulti: Kejuruteraan

Serat gentian panjang yang terhasil dari batang kenaf amat diperlukan oleh industri untuk menghasilkan tekstil dan panel biokomposit. Jentera penuai kenaf sediaada yang diubahsuai dari jentera seperti jentuai tebu, "forage harvesters", "choppers", "reaper binder", "mower conditioner" dan "pedestrian harvester" memotong batang kenaf kepada ukuran pendek, menghancurkannya atau tidak berupaya untuk memotong batang kenaf yang tebal. Justeru itu sebuah jentera penuai kenaf amat diperlukan untuk memotong batang kenaf dalam ukuran yang panjang tanpa merosakkan serat gentian. Buat masa ini proses memotong batang kenaf dilakukan menggunakan tenaga buruh yang memakan masa serta kurang ekonomik.

Sebuah mesin penuai batang kenaf jenis tarik yang berupaya memotong empat baris batang kenaf menerusi sistem pemotong impak berputar telah direkabentuk, dibangunkan dan diuji. Rekabentuk sistem pemotong mesin adalah berdasarkan sudut tepi bilah berkesan (ANE) dan kelajuan putaran memotong. Dalam kajian ini daya pemotongan tentu (SCF) dan tenaga pemotongan tentu (SCE) di ukur dengan mengambilkira sudut tepi bilah (ANE), sudut ricih (SA), sudut tuju bilah (ANA), sudut sadak bilah (ANR), dan luas keratan rentas batang pokok. Sebuah mesin ujian pemotongan impak turut dibina dan diuji di ladang. Penggunaan mesin ujian menunjukkan kelajuan putaran yang dicapai memberikan nilai kilasan terendah. Batang kenaf dari jenis V36 telah digunakan sebagai bahan kajian. Analisis varian bagi nilai SCF dan SCE menunjukkan bahawa kesan kesemua sudut yang dinyatakan adalah signifikan. Berdasarkan ujian julat berbilang Duncan (DMRT), nilai ANE, SA, ANA dan ANR adalah 25°, 40°, 40° dan 40°.

Berdasarkan ujian pemotongan impak, kelajuan pemotongan putaran memberi kesan yang signifikan ke atas nilai kilasan pemotongan. Peningkatan kelajuan putaran dari 308 hingga ke 788 psm akan menurunkan nilai kilasan potongan sebanyak 26.3%. Kelajuan putaran yang dipilih dalam merekabentuk sistem pemotongan adalah 712 psm. Mesin ujian pemotonagn impak berkeupayaan untuk mengerjakan 0.56 ha/8 jam sehari. Kelembapan sampel dari bahagian bawah batang kenaf yang dipotong adalah 70.78% (asas kering).

Pencapaian terbaik mesin adalah pada kelajuan 3-6 km/jam semasa ujian kualiti pemotongan dan berdasarkan ujian julat berbilang Duncan (DMRT). Pada ujian awal, purata nilai keupayaan ladang berkesan (EFC), kecekapan ladang (FE), dan keupayaan bahan (MC) mesin adalah masing-masing 1.68 ha/ 8 jam sehari, 70.6% dan 114.8 ton/8 jam sehari bagi penuaian satu baris (dengan 75 cm jarak tanaman

dan 20 batang/m). Bagi penuaian 2 baris, nilai yang diperolehi adalah masing-masing 3.37 ha/8 jam sehari, 74% dan 241.9 t/8 jam sehari. Jangkaan nilai EFC, FE, dan MC bagi pemotongan 4 baris (dengan 30 cm jarak tanaman dan 10 batang/m) adalah masing-masing 2.92 ha/8 jam sehari, 77% dan 249 t/8 jam sehari pada kelajuan 3-6 km/jam dengan andaian ladang yang bersih dari rumpai dan benih tanaman ditanam pada jarak yang tepat beserta penjagaan yang rapi. Ketinggian maksimum, purata garispusat, purata ketinggian pemotongan dan purata kelembapan batang kenaf ketika penuaian dijalankan adalah masing-masing 3.10 m, 21.8 mm, 20 cm dan 71.8% (asas kering).

Berdasarkan kajian ini keupayaan ladang berkesan maksimum mesin mampu mengatasi 370 pekerja sehari sekiranya dilaksanakan secara manual. Berdasarkan analisis kos, kos penuaian secara manual adalah 32 kali ganda lebih tinggi berbanding kos penuaian secara mekanikal apabila menuai 1,500 hektar tanaman kenaf untuk pengeluaran serat bagi tahun 2010.

#### ACKNOWLEDGMENTS

I wish to express my deepest gratitude to the numerous people who have walked with me along the journey of this thesis. First and foremost I would like to express my deep gratefulness to my supervisor Professor Ir. Dr. Desa Ahmad, the chairman of the supervisory committee for his kind assistance, strong support, critical advice, encouragement, suggestions and direction throughout my research and preparation of this thesis. I would also like to express my gratitude towards Dr. Jamarei Othman and Dr. Khalina Abdan, the members of supervisory committee for their supervision, helpful advice and fruitful discussions that made an invaluable contribution to this dissertation. My appreciation is addressed to Dr. Hadi Suryanto, the former UPM lecturer to encourage and guide me at the onset of design procedure and his valuable advice on cutting system design. I appreciate Associate Professor Dr. Farrokh J. Sharifi (Department of Mechanical and Industrial Engineering, Ryerson University, Toronto, Canada), Professor Dr. Abbas Hemmat (Department of Mechanics of Agricultural Machinery, Isfahan University of Technology, Isfahan, Iran) for their helpful advice and discussion in mechanical design of the present kenaf harvesting machine, and Professor Dr. Ken Giles (Department of Biological and Agricultural Engineering, University of California Davis, USA) for reviewing the research methodology and results of the published full-length paper in Transactions of The ASABE.

This research was funded by the Ministry of Higher Education (MOHE, Malaysia), FRGS Top-Down Project No. 5523501-10201. The author is grateful to Malift Sdn. Bhd (Selangor, Malaysia) for fabricating the prototype cutting machine and the kenaf harvesting machine. My special thankfulness is for Universiti Putra Malaysia (UPM) to support me during my thesis and for granting access to the test field and for providing all the necessary equipment. My special appreciations are extended to Associate Professor Dr. Azmi Yahya, Head of Machinery Design Laboratory for the guidance and cooperation in the cutting tests. My special appreciation is for all staff of the Department of Biological and Agricultural Engineering, UPM who contribute in the completion of this study. Thanks are extended to staff of Institute of Tropical Forestry and Forest Products (INTROP) and Malaysian Agricultural and Research Development Institute (MARDI) for granting us the permission to conduct the field test and evaluation in their plantations. My sincere appreciation also goes to all the people who have helped and supported me.

Last but not the least, my heart-full gratitude and love to my wife and my son, my mother and my father, and siblings whose unconditional support and love has made this dream comes true to me. I certify that a Thesis Examination Committee has met on 25 August 2011 to conduct the final examination of Omid Ghahraei on his thesis entitled "Design and Development of Kenaf Harvesting Machine" in accordance with Universities and University College Act 1971 and the Constitution of the Universiti Putra Malaysia [P.U.(A) 106] 15 March 1998. The Committee recommends that the student be awarded the Doctor of Philosophy.

Members of the Examination Committee are as follows:

#### Wan Ishak bin Wan Ismail, PhD

Professor Faculty of Engineering Universiti Putra Malaysia (Chairman)

#### Shamsuddin bin Sulaiman, PhD

Professor Faculty of Engineering Universiti Putra Malaysia (Internal Examiner)

## **Rimfiel bin Janius, PhD**

Senior Lecturer Faculty of Engineering Universiti Putra Malaysia (Internal Examiner)

## Scott A. Shearer, PhD

Professor Faculty of Engineering University of Kentucky United States of America (External Examiner)

#### NORITAH OMAR, PhD

Associate Professor and Deputy Dean School of Graduate Studies Universiti Putra Malaysia

Date:

This thesis was submitted to the Senate of Universiti Putra Malaysia and has been accepted in fulfilment of the requirements for the degree of **Doctor of Philosophy**. Members of the Supervisory Committee were as follows:

# Desa Ahmad, PhD

Professor Faculty of Engineering Universiti Putra Malaysia (Chairman)

#### Jamarei Othman, PhD

Senior Lecturer Faculty of Engineering Universiti Putra Malaysia (Member)

## Khalina Abdan, PhD

Senior Lecturer Faculty of Engineering Universiti Putra Malaysia (Member)

> HASANAH MOHD GHAZALI, PhD Professor and Dean School of Graduate Studies Universiti Putra Malaysia

Date:

## DECLARATION

I declare that the thesis is my original work except for quotations and citations which have been duly acknowledged. I also declare that it has not been previously and is not concurrently submitted for any other degree at Universiti Putra Malaysia or other institutions.



Zahrar

Date: 25 August 2011

# TABLE OF CONTENT

			Page
DEDICA	TION		ii
ABSTRACT			iii
ABSTRA	K		vi
ACKNOV	WLEDO	GMENTS	ix
APPROV	<b>AL</b>		xi
DECLAR	RATION	3	xiii
LIST OF	TABLI	ES	XX
LIST OF	FIGUR	RES	xxii
LIST OF	ABBRI	EVIATIONS	xxviii
СНАРТЕ	CR		
1	INT	RODUCTION	1
	1.1	Overview	1
	1.2	Brief Review of Kenaf Harvesting Systems	4
	1.3	Problem Statement	5
	1.4	Objectives	6
	1.5	Scope of Study	7
2	LIT	ERATURE REVIEW	8
	2.1	Kenaf Origin, Planting History, and Production	8
	2.2	Kenaf Agronomy Aspects	10
		2.2.1 Kenaf Cultivation	11
		2.2.2 Stem Fibre Yield and Quality	15
	2.3	Research and Activities on Kenaf Plantation in Malaysia	15
	2.4	Kenaf Harvesting	19
		2.4.1 Kenaf Maturity and Harvesting Time	19
		2.4.2 Kenaf Harvesting Methods	20
	2.5	Research on Kenaf Harvesters in Malaysia	30
		2.5.1 TASH (Tractor-Attached Sugarcane Harvester)	30
		2.5.2 4WD Tractor-Mounted Forage Harvester	31
		2.5.3 New Holland Haybine	31
		2.5.4 Small Scale Pedestrian-Type Kenaf Harvester	32
		2.5.5 Small Four-Row Kenaf Harvesting Machine	33
	2.6	Present Unanswered Questions and Untried Methods	34

2.7	Cutting	g Systems and Cutting Mechanisms in Harvesters	35
	2.7.1	Cutter-Bar Type Cutting Mechanism	37
	2.7.2	Rotary Cutting Mechanism	38
	2.7.3	Cutting Mechanism of Choppers (Forage Harvesters)	49
	2.7.4	Chopping Mechanism of Choppers (Forage Harvesters	) 52
2.8	Cutting	g Plant Material	54
	2.8.1	Definition of Cutting	54
	2.8.2	Plant Material Structure	54
	2.8.3	Cutting Device Components (Knife and ountershear)	54
		and Materials	
	2.8.4	Cutting Process	56
	2.8.5	Solid Cut after Compression (Similar to Cutting	59
		Process of Kenaf Stem)	
	2.8.6	Effect of Countershear on Cutting Process	62
	2.8.7	Effective Blade Angles on Cutting Force and	63
		Cutting Energy in a Cutting System	
	2.8.8	Research on Blade Angles and Cutting	66
		Different Crops	
	2.8.9	Cutting Force Concept	67
	2.8.10	Edge Force and Wedge Forces	68
	2.8.11	Support Forces	70
	2.8.12	Impact Cutting (Inertia Cut, or Free Cut)	71
	2.8.13	Force-Displacement Diagram for Cutting	73
	2.8.14	Effective Factors on Cutting Force, Cutting	74
		Energy, and Power in Impact Cutting	
2.9	Summ	ary of the Literature	77
MA'	<b>FERIA</b>	LS AND METHODS	81
3.1	Cutting	g Properties of Kenaf Stems	81
3.2	Cuttin	g System and Cutting Mechanism Selection for	
	Kenaf	Stems	82
3.3	Param	eters Considered in Designing a Cutting	
	Blade	for Harvesting Stems	83
	3.3.1	Cutting Knife Angles	83
	3.3.2	Cutting Knife Dimension	83
	3.3.3	Cutting Knife Speed	86
	3.3.4	Cutting Knife Material	86

3

3.4	Techniques for Evaluating the Parameters	86
	Considered in Designing a Cutting Blade	
	for Harvesting Stems	
	3.4.1 Evaluating Techniques on Cutting Knife	86
	Angles and Dimensions	
	3.4.2 Evaluating Technique on Cutting Knife Speed	90
3.5	Kenaf Harvesting Machine Configuration	92
3.6	Cutting System Design	93
	3.6.1 Cutting Force and Cutting Energy Requirement	94
	3.6.2 Cutting Time Used for One Stem	96
	3.6.3 Time Needed for Disk to Move from One	97
	Stem to Another	
	3.6.4 Energy Saved in Cutting Disk and its Mass	99
	Requirement	
	3.6.5 Final Power Requirement for Cutting System	101
3.7	Fabrication of Rotary Cutting Disk	102
	3.7.1 Impact Cutting Speed	102
	3.7.2 Dimension of Rotary Cutting Disk	106
3.8	Fabrication of Cutting Blades	107
3.9	Cutting System Test in Workshop	108
3.1	0 Evaluation of Prototype Cutting Machine in the Field	109
3.1	1 Gathering System	112
	3.11.1 Power Requirement for Gathering System of	118
	the Kenaf Harvesting Machine	
3.12	2 Chassis, Drawbar, Offset Connecting System, and	119
	Wheels	
	3.12.1 Power Requirement for Pulling the Kenaf	120
	Harvesting Machine (Drawbar Power)	
3.1	3 Cutting Height Adjustment System	121
3.1	4 Power Transmission System	123
	3.14.1 Total Power Requirement for the kenaf Harvesting Machine and Tractor	128
	Selection for Field Operation	
3.1	5 Field Tests of Kenaf Harvesting Machine	131
	3.15.1 Cutting Quality Test at Varying	133
	Harvesting Speeds	
	3.15.2 Field Capacity Tests	134

	3.16 Economic Cost Analysis	136
	3.16.1 Fixed Costs (Ownership Costs)	138
	3.16.2 Variable Costs (Operating Costs)	140
	3.16.3 Ownership and Operating Costs Calculation	143
	3.17 Advantages and Novelty of the Present Kenaf	149
	Harvesting Machine	
	3.17.1 Advantages and Novelty of the New Cutting System	149
	3.17.2 Benefits and Novelty of New Blade Design	150
4	RESULTS AND DISCUSSION	152
2	4.1 Effect of Knife Edge Angle on the Specific Cutting	152
	Force and Energy	
4	4.2 Effect of Knife Approach Angle, Shear Angle,	156
	and Knife Rake Angle on the Specific Cutting	
	Force and Energy	
2	4.3 Effect of Stem Cross-Sectional Area on	162
	Specific Cutting Force and Energy	
4	4.4 Specific Cutting Torque Requirement for	164
	Impact Cutting	
2	4.5 Field Capacity of the Experimental Prototype Cutting Machine	165
2	4.6 Main Technical Parameters of the Developed Kenaf	166
	Harvesting Machine	
2	4.7 Field Tests of Kenaf Harvesting Machine	166
	4.7.1 Cutting Quality Test Results	170
	4.7.2 Field Capacity Test Results	177
	4.8 Economical Cost Analysis	183
5	CONCLUSION AND RECOMMENDATIONS	188
5	5.1 Conclusion	188
(C) <sup>5</sup>	5.2 Recommendations for Future Studies	194
REFERENC	CES	197
APPENDICES		
A	Drawings from Different Views of the	207
]	Kenaf Harvesting Machine (Dimensions in mm)	

В	Hydraulic Cylinder Force Vector and its Exact Location from Side View by Solidworks 2009	209
	Software (Dimensions in mm)	
С	2WD John Deere 6405 Tractor Gears No. and Related	210
	Speeds Used in the Field Tests and Speed Calibration	

# BIODATA OF STUDENT LIST OF PUBLICATIONS

211 213

